

# **Implementation of a constitutive model for masonry shells as a stand-alone subroutine**

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# 1 Introduction

This report deals with the implementation of a macro-model for the out-of-plane behavior of masonry structures. The composite plasticity model proposed in Lourenço (2000) combines anisotropic elastic behavior with anisotropic plastic behavior and is formulated in modern algorithmic plasticity concepts. These include implicit Euler backward return mapping schemes and consistent tangent operators for all regimes of the model. The yield criterion combines the advantages of modern plasticity concepts with a powerful representation of anisotropic material behavior, which incorporates different hardening/softening behavior along each material axis. The model is thus capable of reproducing independent (in the sense of completely diverse) elastic and inelastic behavior along a prescribed set of material axes. The energy-based regularization technique, which is employed to obtain objective results with respect to mesh refinement, resorts then to four different fracture energies.

The numerical implementation of the model is evaluated by means of a comparison between numerical results and experimental results for the case of masonry panels with out-of-plane loading. The performance of the model is evaluated by means of a comparison with experimental results in reinforced masonry shells.

## 2 Masonry Out-of-Plane Behavior

### 2.1 Uniaxial Behavior

The flexural strength of masonry has been mostly investigated in relation to the resistance of wall panels to wind loads. The flexural strength is, of course, different for bending perpendicular or parallel to the bed joints, see Figure 2.1, being normally several times larger when bending leads to failure in a plane perpendicular to the bed joints. In general, experimental results have been concerned solely with measurements of the flexural strength and mostly for the case when the plane of failure occurs parallel to the bed joints, while elastic and inelastic properties have usually been ignored.

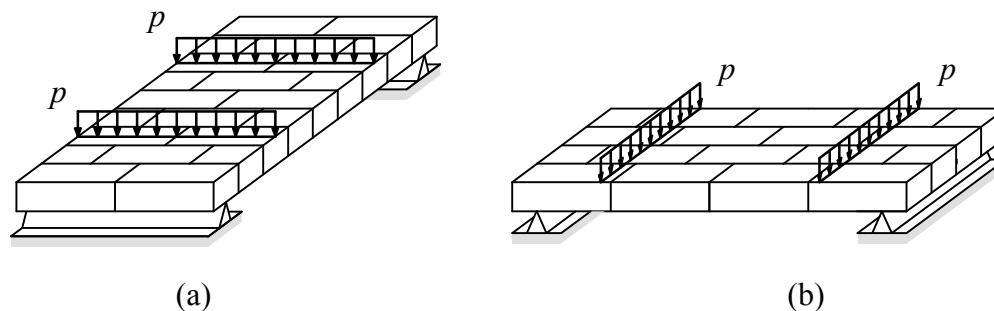


Figure 2.1 - Four-point bending test in two different directions: (a) plane of failure parallel to the bed joints – vertical bending or bending normal to the bed joints – and (b) plane of failure perpendicular to the bed joints – horizontal bending or bending parallel to the bed joints

In the case of bending leading to failure in a plane parallel to the bed joints (“vertical bending”), failure is generally caused by the relatively low tensile bond strength between the bed joints and the unit, see Figure 2.2a. In masonry with low strength units and greater tensile bond strength between the bed joints and the unit, e.g. high-strength mortar and units with numerous small perforations, which produce a dowel effect, failure may occur as a result of stresses exceeding the unit tensile strength. In any case, the typical moment-curvature relation is linear up to 70%-85% of the failure load, Lawrence (1983)

and van der Pluijm et al (1995), see Figure 2.3a. Beyond this level, micro-cracking starts to occur, associated with unloading (“softening”) of the extreme tensile fibers.

For bending leading to failure in a plane perpendicular to the bed joints (“horizontal bending”), two different types of failure are common, depending on the relative strength of joints and units, see Figure 2.2b. In the first type of failure cracks zigzag through head and bed joints. The post-peak response of the specimen is governed by the fracture energy of the head joints and the friction behavior of bed joints. In the second type of failure cracks run almost vertically through the units and head joints. The post-peak response is governed by the fracture energy of the units and head joints. In both types of failure, the typical moment-curvature relation indicates a sudden decrease of stiffness before non-linear behavior starts to occur, see Figure 2.3b. This kind of behavior was originally noticed by Ryder (1963). Lawrence (1983) attributed this behavior to the gradual decrease of stiffness of the head joints, i.e. the softening behavior, which has been confirmed by van der Pluijm et al (1995). It is noted that the above holds true only in the general case of units with a tensile strength substantially larger than the tensile strength of the head joints.

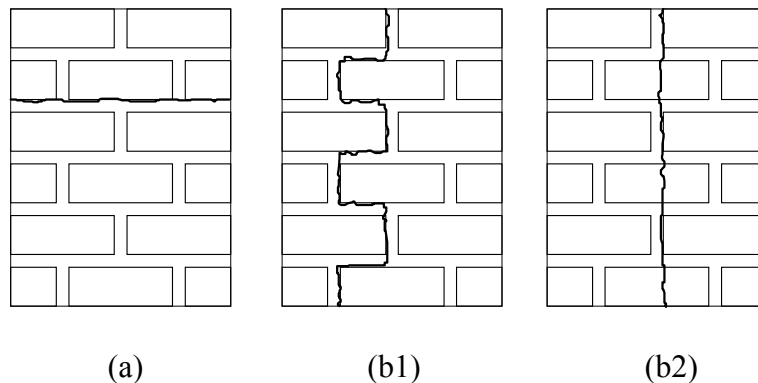


Figure 2.2 - Possible failure modes for masonry subjected to bending along the material axes. Failure in a plane parallel to the bed joints: (a) “debonding”. Failure in a plane perpendicular to the bed joints: (b1) “toothed” and (b2) “splitting”

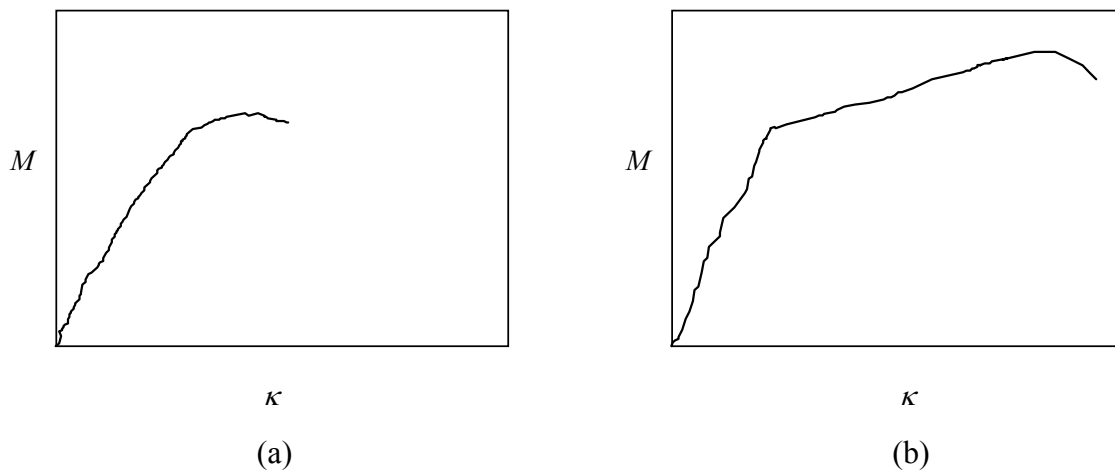
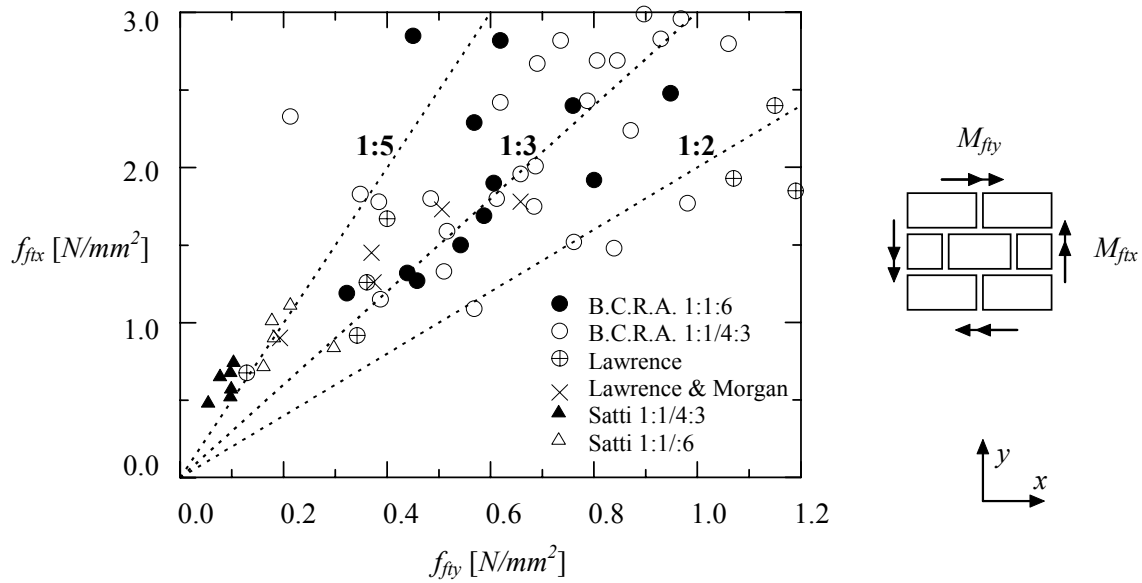


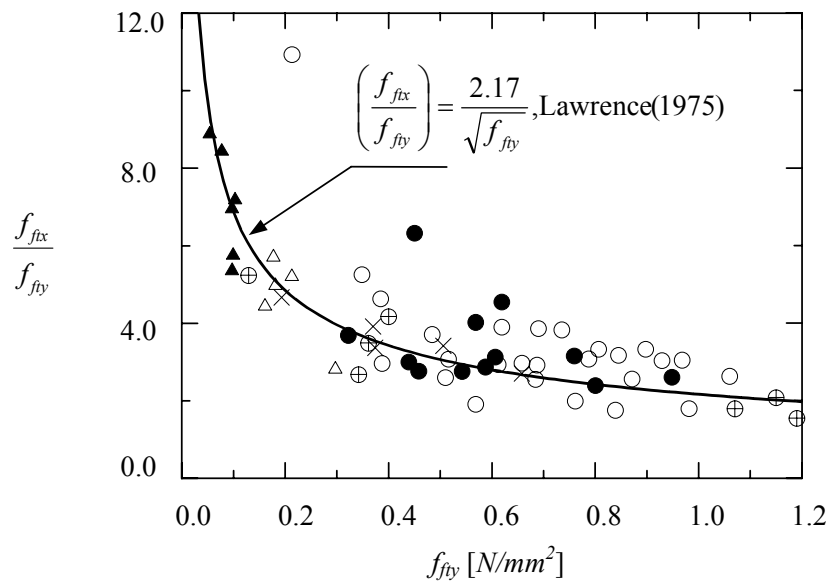
Figure 2.3 - Typical moment-curvature ( $M$ - $\kappa$ ) diagrams for masonry in bending such that failure occurs in a plane (a) parallel and (b) perpendicular to the bed joints

The ratio of flexural strength along the two material axes varies significantly as can be seen in Figure 2.4a. Nevertheless, a definite trend seems to be distinguished in the orthogonal strength ratio, which decreases markedly with an increasing flexural bond strength, see Figure 2.4b and also Baker (1979). It is noted that, a large scatter occurs and such a simplified assumption is hardly acceptable. The scatter is partly caused by:

- Different specimen and load configurations used to determine flexural strengths;
- The variability of the masonry constituents and workmanship. Ideally, each point should be established by averaging a convenient number of tests;
- Different flexural strengths of the units used. Since failure in bending parallel to the bed joints often occurs by unit breaking after degradation of the head joints strength, this parameter is important. This cannot be distinguished by such an empirical curve but a theoretical relationship that takes into account the flexural strength of the units can be found in Baker (1979).



(a)



(b)

Figure 2.4 - Flexural tensile strength along the two material axes, Hendry (1990):  
(a) strength along x-axis vs. strength along y-axis; (b) strength ratio vs. strength along y-axis

## 2.2 Biaxial Behavior

There is relatively little knowledge about the flexural strength at different angles to the bed joints or about the interaction relationships for biaxial flexural tension. Losberg and Johansson (1969) present, probably, the first paper about four-point bending tests with masonry beams featuring different orientations with respect to the bed joints. This is a very complete article and a remarkable work for such early period. Satti and Hendry (1973) reported that the flexural strength at 45 degrees to the bed joints was approximately one half to a third of the strength in horizontal bending. In contrast, Cajdert and Losberg (1973) found that the diagonal flexural strength was higher than both the vertical and horizontal strengths. However, these early works were based on very few tests and there was a wide scatter of results.

Recently researchers have tried to gather the necessary experimental insight in biaxial bending, Gazzola and Drysdale (1986), Guggisberg and Thürlimann (1990) and van der Pluijm et al (1995). Gazzola and Drysdale (1986) and van der Pluijm et al (1995) performed similar tests, resorting to four-point bending masonry beams loaded in different orientations with respect to the material axes, but only van der Pluijm et al (1995) was able to measure and report moment-curvature diagrams. Guggisberg and Thürlimann (1990) resorted to a more complex test set-up capable of applying a general combination of bending moments. Figure 2.5 illustrates possible flexure tensile failure modes of masonry subject to out-of-plane loading.

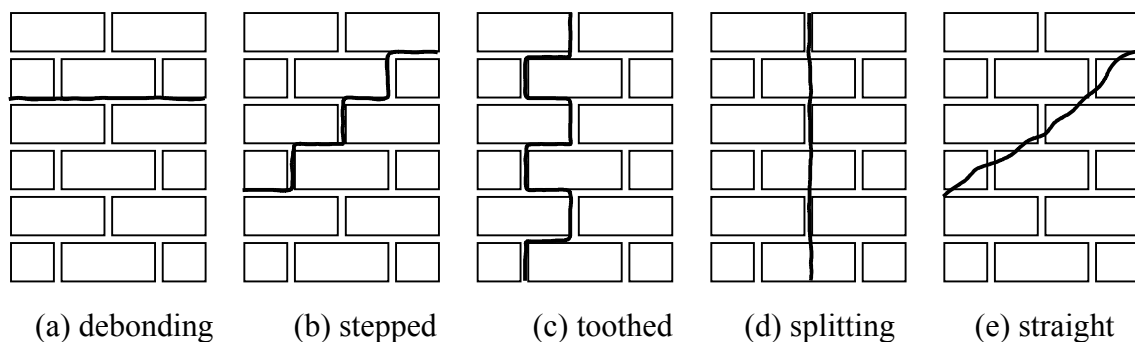


Figure 2.5 - Possible flexure failure modes

Being masonry an anisotropic material, its complete out-of-plane behavior must be established in terms of, either, the three moment-components ( $M_{xx}$ ,  $M_{yy}$  and  $M_{xy}$ ), or, the



principal moments and one angle  $\theta$ , which measures the orientation of the principal axes with respect to the material axes. It is noted that, in the general case of shell analysis, the material behavior must be described by five stress components (the usual assumption of thin structures assumes the out-of-plane normal stress to remain zero) and a yield criterion established solely in terms of principal moments is inadequate.

The experimental failure patterns of Gazzola and Drysdale (1986), in hollow concrete block masonry, are summarized in Figure 2.6. Five tests were carried out, for each orientation of the principal bending moment  $\theta$ . Failure occurred, generally, in a combination of tension and shear, except for some cracking through the units observed for bending parallel to the bed joints. The average and standard deviation of the five tests, for each orientation of loading, are represented in Figure 2.7. It is observed that, in this particular testing program, the flexural strength for bending parallel to the bed joints is 2.5 times larger than the flexural strength for bending normal to the bed joints.

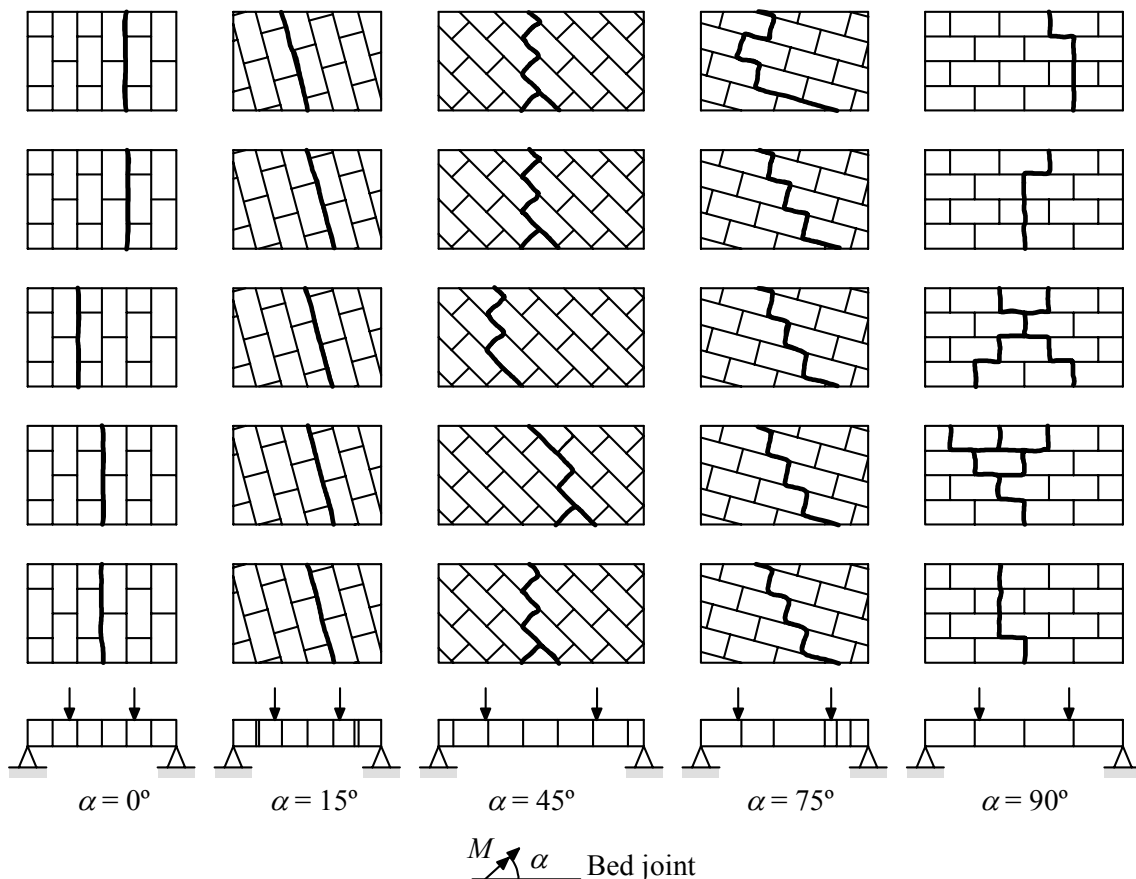


Figure 2.6 – Failure patterns in masonry wallettes, Gazzola and Drysdale (1986)

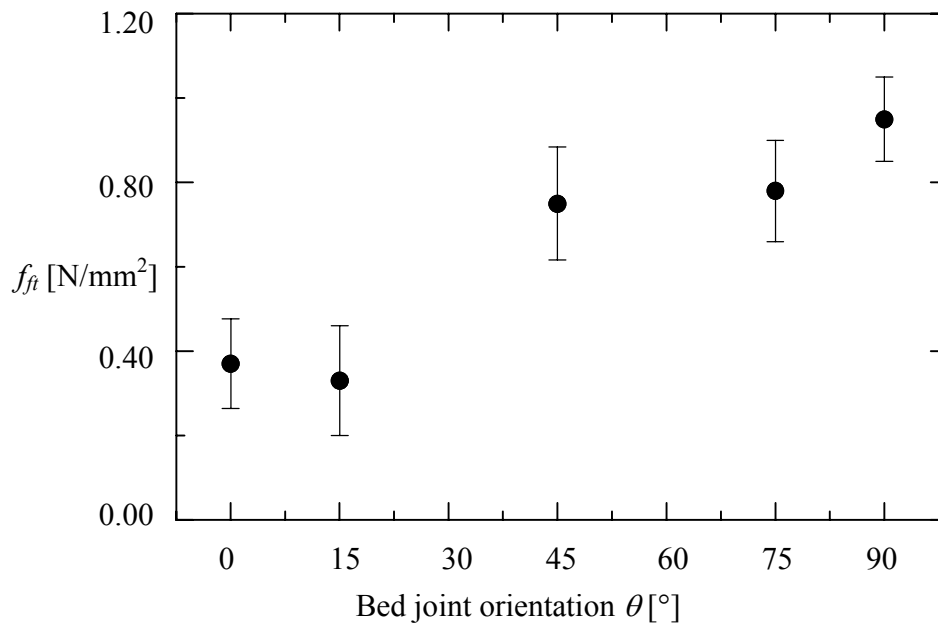


Figure 2.7 - Flexural strength for different orientations of loading,  
Gazzola and Drysdale (1986)

The results from van der Pluijm et al (1995), in solid brick clay masonry and calcium silicate block masonry, are summarized in Figure 2.8 and Figure 2.9 by the average and standard deviation of the tests, for each orientation of loading. Twelve tests were carried out, for each orientation of the principal bending moment  $\theta$ . Failure occurred, generally, in a combination of tension and shear in the head and bed joints, except for loading making an angle of 70 degrees with the bed joints, where cracking through the units was also observed. It can be seen that, in this particular testing program, the flexural strength for bending parallel to the bed joints is approximately 4.0 times larger than the flexural strength for bending normal to the bed joints.

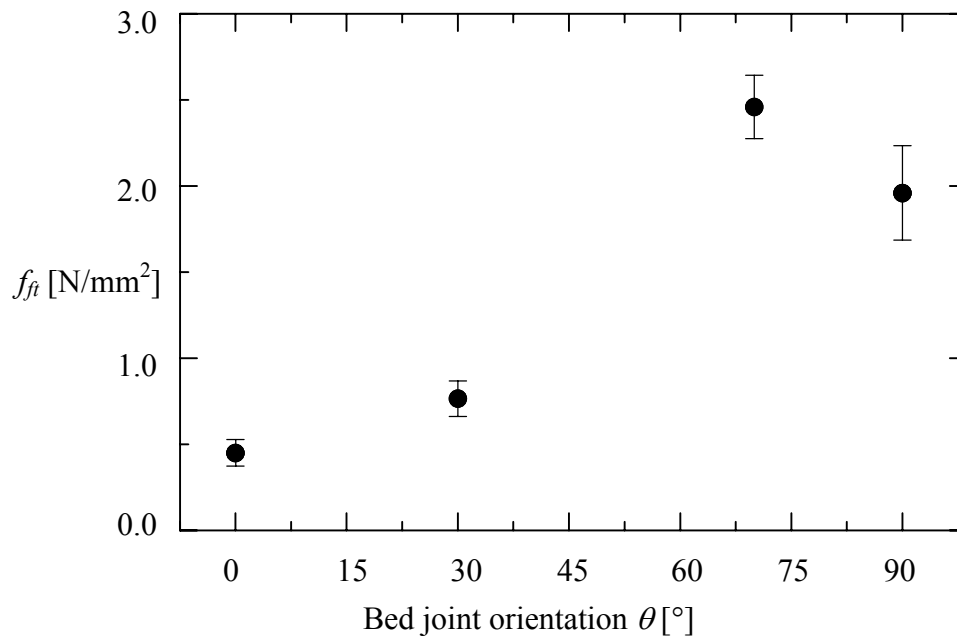


Figure 2.8 - Flexural strength for different orientations of loading in solid brick clay masonry, van der Pluijm et al (1995)

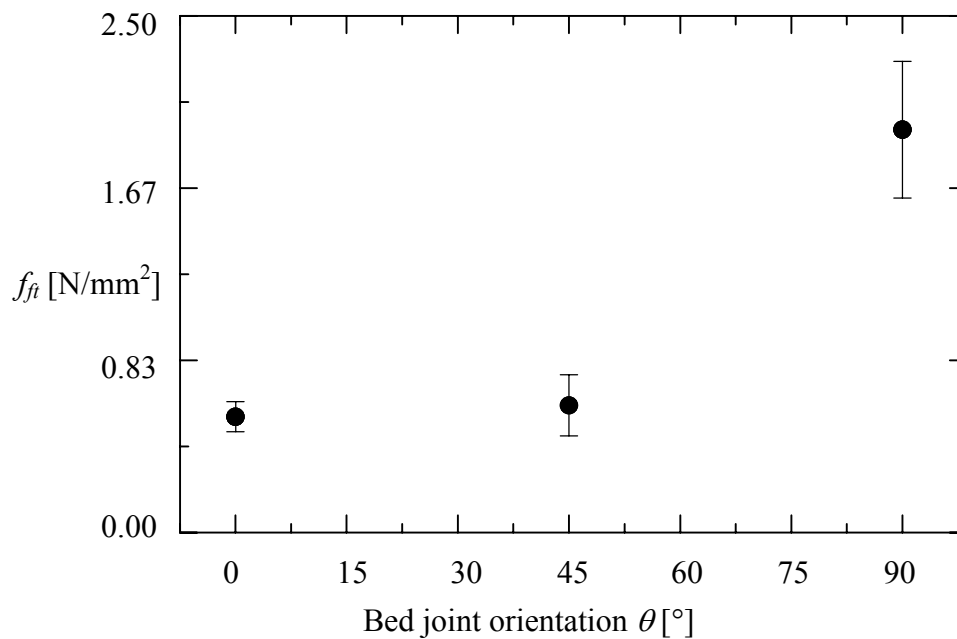


Figure 2.9 - Flexural strength for different orientations of loading in solid block calcium silicate masonry, van der Pluijm et al (1995)

### 3 Description of the Anisotropic Continuum Model

In this study, the problem of modeling laminar masonry structures (plates and shells with one dimension substantially smaller than the other two dimensions) is addressed. A typical hypothesis in this type of elements is “zero normal stress”. This hypothesis states that the normal stress component perpendicular to the plane of the structure equals zero, and simplifies material modeling to a great extent.

#### 3.1 Plates, shells and finite elements

Plates and shells are but a particular form of a three-dimensional solid, the treatment of which presents no theoretical difficulties. Owing to the symmetry of the stress and strain tensors, it is normal to collect the tensors components, see Figure 3.1, in vectors as

$$\begin{cases} \{\sigma\} = \{\sigma_x, \sigma_y, \sigma_z, \tau_{xy}, \tau_{yz}, \tau_{xz}\}^T \\ \{\varepsilon\} = \{\varepsilon_x, \varepsilon_y, \varepsilon_z, \gamma_{xy}, \gamma_{yz}, \gamma_{xz}\}^T \end{cases} \quad (3.1)$$

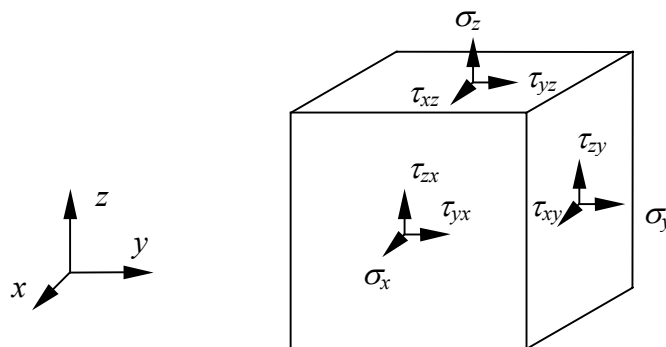


Figure 3.1 - Stress components in a three-dimensional body

However, the thickness of plates and shells (denoted in the following as  $t$ ) is small when compared with other dimensions, and complete three-dimensional numerical treatment would be, in general, not only costly but in addition could lead to serious equation conditioning problems. To ease the solution several classical assumptions



regarding the behavior of such structures were introduced. Clearly such assumptions resulted in a series of approximations, which led, basically, to a thin plate theory and a thick plate theory. Here, no discussion is given regarding the approximations involved or the range of validity of the theories. For a comprehensive review of plate and shell bending approximations and the finite element implementation the reader is referred to Zienkiewicz and Taylor (1991).

Here, the finite element adopted to reduce the degrees-of-freedom in a complete three-dimensional analysis, is the curved shell element degenerated from a three-dimensional formulation. This element, originally proposed by Ahmad et al (1970) for the linear analysis of moderately thick shells, has been extensively used for the geometrical and non-linear analysis of shell structures. Typical characteristics of this element are the two hypotheses on which the degeneration is based: “straight normals” and “zero normal stress”. The first hypothesis assumes that the normals to the mid-plane of the element remain straight after deformation, but not necessarily perpendicular to the mid-plane. The second hypothesis, already introduced above, states that the normal stress component perpendicular to the mid-plane equals zero, and the element formulation has been obtained ignoring the strain energy resulting from this stress. Assuming that the local z-axis represents the normal to the mid-plane, the five stress components left are  $\sigma_x$ ,  $\sigma_y$ ,  $\tau_{xy}$ ,  $\tau_{yz}$  and  $\tau_{xz}$ , see Figure 3.2.

Five degrees of freedom are defined for each element node: three translations and two rotations, see Figure 3.3. The definition of the independent translations and rotations includes the influence of shear deformation. The rotations are not coupled to the gradient of the mid-plane. In this study, two by two Gauss integration in the plane and seven-point Simpson integration in the thickness direction are used.

As the behavior of plates and shells is, typically, two-dimensional, it is possible to adopt a yield criterion developed for plane stress conditions enhanced with two new stress components. Therefore, the adopted yield criterion is based on the plane stress anisotropic yield criterion of Lourenço (1996), which includes a Hill type criterion for compression and a Rankine type criterion for tension, see Figure 3.4. The word type is used here because the yield criteria adopted are close to the original yield criteria, even if they represent solely a fit of experimental results.

In the present implementation, there is no difference between the formulation for a complete three-dimensional stress-state and the formulation for shells, which is quite convenient for general purpose finite element programs. Therefore, in the present study, all the formulation will be given in the full six components stress-strain space. It suffices to use a Young's modulus  $E_z$  equal to zero, to retrieve the traditional five component formulation. The full model implementation is given in Annex A in the format of a Fortran file.

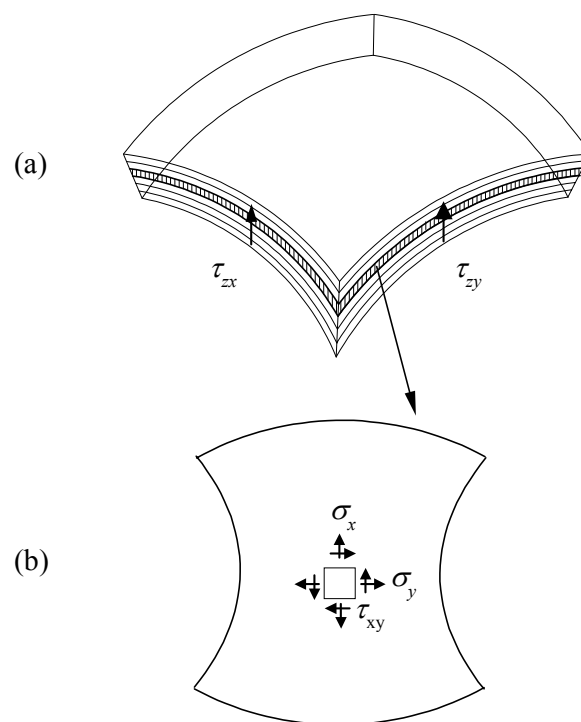


Figure 3.2 - Thin shells: (a) layered shell with five stress components; (b) layer, essentially, in plane stress conditions

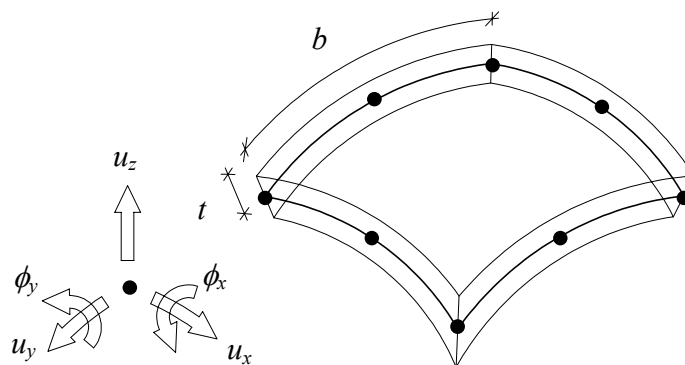


Figure 3.3 - Curved shell element (applicable for thin shells with  $t \ll b$ )

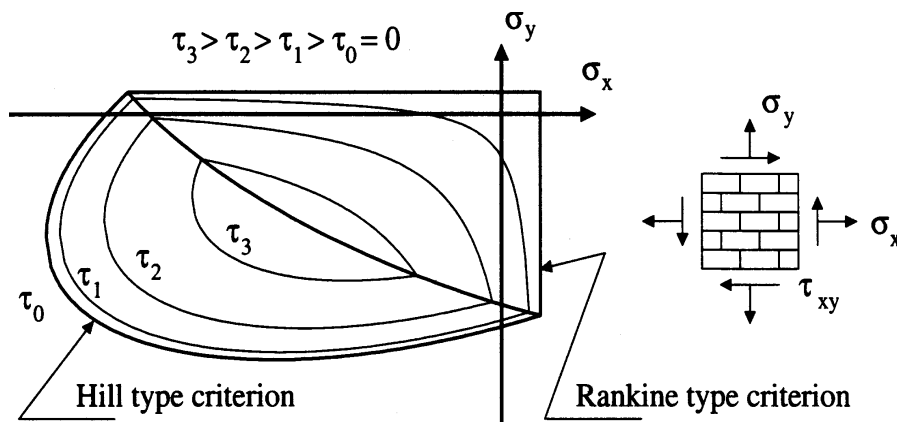


Figure 3.4 - Adopted plane stress composite yield criterion with iso-shear stress lines,  
Lourenço (1996)

### 3.2 The Rankine Type Criterion

An adequate formulation of the Rankine criterion is given by a single function, which is governed by the first principal stress and one yield value  $\bar{\sigma}_t$  that describes the softening behaviour of the material as, see Feenstra and de Borst (1995),

$$f_1 = \frac{\sigma_x + \sigma_y}{2} + \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2} - \bar{\sigma}_t(\kappa_t) \quad (3.2)$$

where the scalar  $\kappa_t$  controls the amount of softening. This expression can be rewritten as

$$f_1 = \frac{(\sigma_x - \bar{\sigma}_t(\kappa_t)) + (\sigma_y - \bar{\sigma}_t(\kappa_t))}{2} + \sqrt{\left(\frac{(\sigma_x - \bar{\sigma}_t(\kappa_t)) - (\sigma_y - \bar{\sigma}_t(\kappa_t))}{2}\right)^2 + \tau_{xy}^2} \quad (3.3)$$

where coupling exists between the stress components and the yield value. Setting forth a Rankine type criterion for an anisotropic material, with different tensile strengths along the  $x, y$  directions, is now straightforward if eq. (3.3) is modified to

$$f_1 = \frac{(\sigma_x - \bar{\sigma}_{tx}(\kappa_t)) + (\sigma_y - \bar{\sigma}_{ty}(\kappa_t))}{2} + \sqrt{\left(\frac{(\sigma_x - \bar{\sigma}_{tx}(\kappa_t)) - (\sigma_y - \bar{\sigma}_{ty}(\kappa_t))}{2}\right)^2 + \alpha \tau_{xy}^2} \quad (3.4)$$

where the parameter  $\alpha$ , which controls the shear stress contribution to failure, reads

$$\alpha = \frac{f_{tx} f_{ty}}{\tau_{u,t}^2} \quad (3.5)$$

Here,  $f_{tx}$ ,  $f_{ty}$  and  $\tau_{u,t}$  are, respectively, the uniaxial tensile strengths in the  $x$ ,  $y$  directions and the pure shear strength. Note that the material axes are now fixed with respect to a specific frame of reference. Thus, it shall be assumed that all stresses and strains for the elastoplastic algorithm are given in the material reference axes, see also Section 3.4.

Eq. (3.4) can be recast in a matrix form as

$$f_1 = (\frac{1}{2} \{\xi\}^T [P_t] \{\xi\})^{1/2} + \frac{1}{2} \{\pi\}^T \{\xi\} \quad (3.6)$$

where the projection matrix  $[P_t]$  reads

$$[P_t] = \begin{bmatrix} \frac{1}{2} & -\frac{1}{2} & 0 & 0 & 0 & 0 \\ -\frac{1}{2} & \frac{1}{2} & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 2\alpha & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} \quad (3.7)$$

the projection vector  $\{\pi\}$  reads

$$\{\pi\} = \{1 \ 0 \ 0 \ 0 \ 0 \ 0\}^T \quad (3.8)$$

the reduced stress vector  $\{\xi\}$  reads



$$\{\xi\} = \{\sigma\} - \{\eta\} \quad (3.9)$$

the stress vector  $\{\sigma\}$ , as given in eq. (3.1), reads

$$\{\sigma\} = \{\sigma_x \quad \sigma_y \quad \sigma_z \quad \tau_{xy} \quad \tau_{yz} \quad \tau_{xz}\}^T \quad (3.10)$$

and the back stress vector  $\{\eta\}$  reads

$$\{\eta\} = \{\sigma_{tx}(\kappa_t) \quad \sigma_{ty}(\kappa_t) \quad 0 \quad 0 \quad 0 \quad 0\}^T \quad (3.11)$$

Exponential tensile softening is considered for both equivalent stress-equivalent strain diagrams, with different fracture energies ( $G_{fx}$  and  $G_{fy}$ ) for each yield value, which read

$$\bar{\sigma}_{tx} = f_{tx} \exp\left(-\frac{hf_{tx}}{G_{fx}} \kappa_t\right) \quad \text{and} \quad \bar{\sigma}_{ty} = f_{ty} \exp\left(-\frac{hf_{ty}}{G_{fy}} \kappa_t\right) \quad (3.12)$$

where the standard equivalent length  $h$  is related to the element size, Bazant and Oh (1983).

A non-associated plastic potential  $g_1$

$$g_1 = (\frac{1}{2} \{\xi\}^T [P_g] \{\xi\})^{1/2} + \frac{1}{2} \{\pi\}^T \{\xi\} \quad (3.13)$$

is considered, where the projection matrix  $[P_g]$  represents the original Rankine plastic flow, i.e.  $\alpha = 1$  in eq. (3.7). The inelastic behaviour is described by a strain softening hypothesis given by the maximum principal plastic strain  $\varepsilon_t^p$  as

$$\dot{\kappa}_t = \dot{\varepsilon}_1^p = \frac{\dot{\varepsilon}_x^p + \dot{\varepsilon}_y^p}{2} + \frac{1}{2} \sqrt{(\dot{\varepsilon}_x^p - \dot{\varepsilon}_y^p)^2 + (\dot{\gamma}_{xy}^p)^2} \quad (3.14)$$

which reduces to the particularly simple expression

$$\kappa_i = \dot{\lambda}_i \quad (3.15)$$

### 3.3 The Hill Type Criterion

The simplest yield criterion that features different compressive strengths along the two material axes is a rotated centred ellipsoid in the full stress space. The expression for such a quadric can be written as

$$f_2 = \frac{\bar{\sigma}_{cy}(\kappa_c)}{\bar{\sigma}_{cx}(\kappa_c)} \sigma_x^2 + \beta \sigma_x \sigma_y + \frac{\bar{\sigma}_{cx}(\kappa_c)}{\bar{\sigma}_{cy}(\kappa_c)} \sigma_y^2 + \gamma(\tau_{xy}^2 + \tau_{yz}^2 + \tau_{xz}^2) - \bar{\sigma}_{cx}(\kappa_c) \bar{\sigma}_{cy}(\kappa_c) = 0 \quad (3.16)$$

where  $\bar{\sigma}_{cx}(\kappa_c)$  and  $\bar{\sigma}_{cy}(\kappa_c)$  are, respectively, the yield values along the material axes  $x$  and  $y$ .  $\beta$  and  $\gamma$  values are additional material parameters that determine the shape of the yield criterion. The parameter  $\beta$  controls the coupling between the normal stress values, i.e. rotates the yield criterion around the shear stress axis, and must be obtained from one additional experimental test, e.g. biaxial compression with a unit ratio between principal stresses. The parameter  $\gamma$ , which controls the shear stresses contributions to failure, can be obtained from

$$\gamma = \frac{f_{cx} f_{cy}}{\tau_{u,c}^2} \quad (3.17)$$

where  $f_{cx}$ ,  $f_{cy}$  and  $\tau_{u,c}$  are, respectively, the uniaxial compressive strengths in the  $x$ ,  $y$  directions and a fictitious pure shear in compression.

For the purpose of numerical implementation, it is convenient to recast this yield criterion in a matrix form as

$$f_2 = (\frac{1}{2} \{\sigma\}^T [P_c] \{\sigma\})^{1/2} - \bar{\sigma}_c(\kappa_c) \quad (3.18)$$

where the projection matrix  $[P_c]$  reads

$$[P_c] = \begin{bmatrix} 2\frac{\bar{\sigma}_{cy}(\kappa_c)}{\bar{\sigma}_{cx}(\kappa_c)} & \beta & 0 & 0 & 0 & 0 \\ \beta & 2\frac{\bar{\sigma}_{cx}(\kappa_c)}{\bar{\sigma}_{cy}(\kappa_c)} & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 2\gamma & 0 & 0 \\ 0 & 0 & 0 & 0 & 2\gamma & 0 \\ 0 & 0 & 0 & 0 & 0 & 2\gamma \end{bmatrix} \quad (3.19)$$

the yield value  $\bar{\sigma}_c$  is given by

$$\bar{\sigma}_c(\kappa_c) = \sqrt{\bar{\sigma}_{cx}(\kappa_c) \bar{\sigma}_{cy}(\kappa_c)} \quad (3.20)$$

and the scalar  $\kappa_c$  controls the amount of hardening and softening.

The inelastic law adopted comprehends parabolic hardening followed by parabolic/exponential softening for both equivalent stress-equivalent strain diagrams, with different compressive fracture energies ( $G_{fcx}$  and  $G_{fcy}$ ) along the material axes, see Lourenço (1996). The problem of mesh objectivity of the analyses with strain softening materials is a well debated issue, at least for tensile behaviour, and the stress-strain diagram must be adjusted according to an equivalent length  $h$  to provide an objective energy dissipation, see Feenstra and de Borst (1996).

An associated flow rule and a work-like hardening/softening hypothesis are considered. This yields

$$\dot{\kappa}_c = \frac{1}{\bar{\sigma}_c} \{\sigma\}^T \{\dot{\varepsilon}^p\} = \dot{\lambda}_c \quad (3.21)$$

### 3.4 Orientation of the Material Axes

For the sake of simplicity, the formulation of the plasticity model was presented based on the assumption that the principal axes of anisotropy coincided with the frame of



reference (local or global) for stresses and strains in finite element computations. Since this is not necessarily the case, such non-alignment effects must be taken into account.

Two different approaches can be followed. In the first approach, with each call to the plasticity model, stresses, strains and, finally, consistent tangent stiffness matrices must be rotated into and out of the material frame of reference, respectively, as pre- and post-processing. In the second approach, before the analysis begins, the elastic stiffness matrix  $[D]$ , the projection matrices  $[P_t]$ ,  $[P_g]$  and  $[P_c]$ , and the projection vectors  $\{p\}$  and  $\{h\}$  must be rotated from the material frame of reference into the global frame of reference at each quadrature point, eliminating the need of all subsequent rotation operations. The drawback of the latter approach is that the matrices then lose their sparse nature, resulting in less clear algorithms. For this reason, the plasticity model is implemented employing the former option.

## 4 Validation

### 4.1 McMaster University hollow concrete block panels

The first series of panels analysed consists of hollow concrete block masonry. The tests were carried out by Gazzola *et al* (1985) and are denoted by W. The inelastic properties of the composite material are obtained from Gazzola *et al* (1985), see Table 4.1 and Table 4.2. Only tensile inelastic behaviour is considered in the analysis and the fracture energy values indicated include already the size  $h$  of the element,  $g_f = G_f / h$ . The elastic properties were not available and had to be estimated. The Young's modulus in the vertical direction has been calculated from the masonry compressive strength according to Hendry (1990), the elastic orthotropy is assumed to equal two and the adopted in plane Poisson's ratio is 0.2. In order to remove the effect of the  $z$ -axis, the following approximately zero values were assumed:  $E_z = 1.d-3$ ,  $\nu_{xz} = 1.d-7$  and  $\nu_{yz} = 1.d-7$ .

Table 4.1 - Material properties for McMaster University panels

$E_x$	$E_y$	$E_z$	$\nu_{xy}$	$\nu_{yz}$	$\nu_{xz}$	$G_{xy}$	$G_{yz}$	$G_{xz}$
5000	10000	1.d-3	0.2	1.d-7	1.d-7	3125	3125	3125
$N/mm^2$	$N/mm^2$	$N/mm^2$	-	-	-	$N/mm^2$	$N/mm^2$	$N/mm^2$

Table 4.2 - Material properties for McMaster University panels

Tension regime				
$f_{tx}$	$f_{ty}$	$\alpha$	$g_{fx}$	$g_{fy}$
0.94	0.35	0.4	0.0012	0.00008
$N/mm^2$	$N/mm^2$	-	$N/mm^2$	$N/mm^2$

Only panel WP1, see Figure 4.1, was analysed with the composite plasticity model. The panel was loaded until failure with increasing out-of-plane uniform pressure  $p$ . This panel was loaded, previously to the application of the out-of-plane loading, with an in-plane confining pressure of  $0.2 N/mm^2$ , which was kept constant until failure of the

specimen due to the pressure  $p$ . For each configuration, three different tests have been carried out and the results reported in Gazzola *et al* (1985) represent the average of the tests.

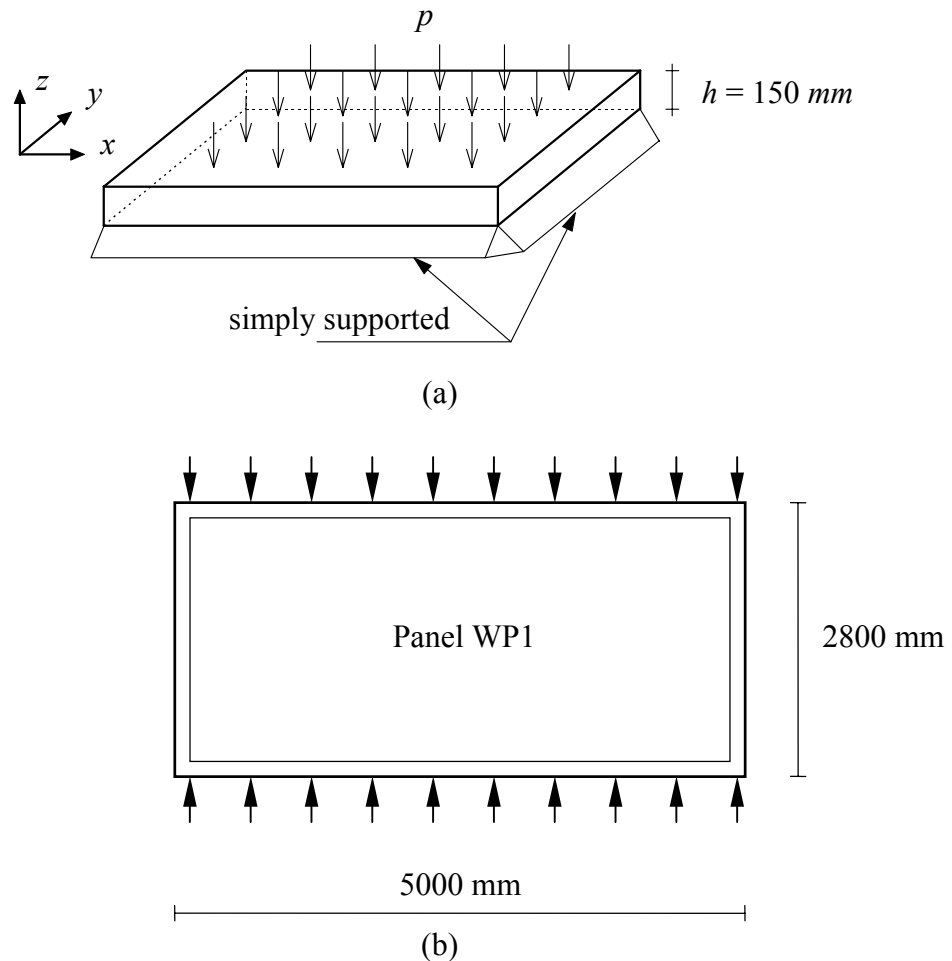
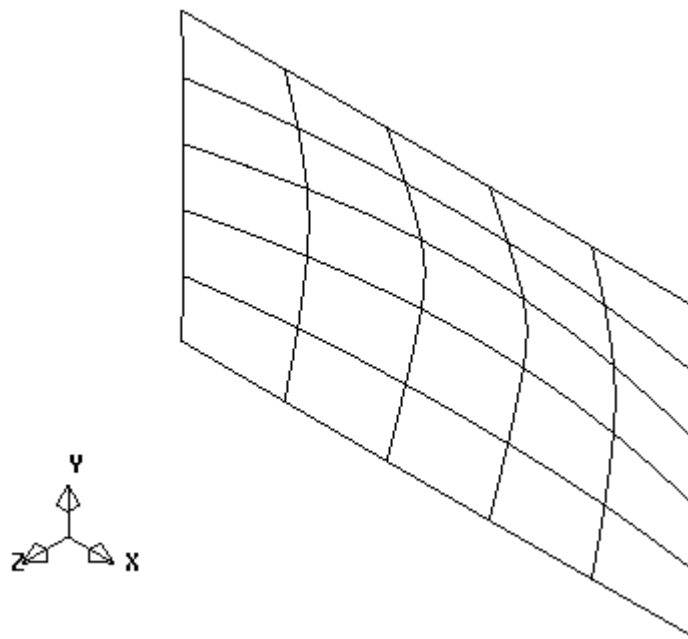


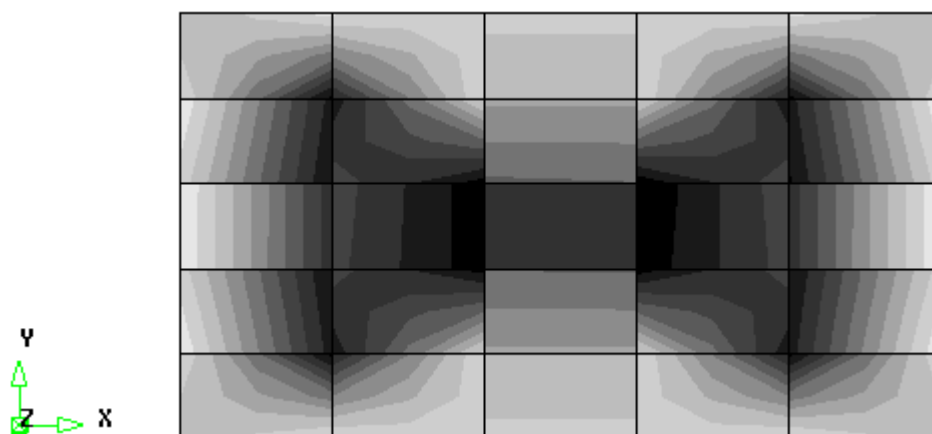
Figure 4.1 - Panels with constant out-of-plane pressure  $p$ , Gazzola *et al* (1985):  
(a) out-of-plane loading; (b) geometry and in-plane loading.

For the numerical analyses eight-noded degenerated shell elements with two by two in-plane Gauss integration and seven-point Simpson through thickness integration are used. Due to the double symmetry of the structure only one quarter (or one half) of the structure needs to be modelled. Nevertheless, for easiness in the post-processing and to obtain more clear pictures, the full structure is modelled. For this purpose, a regular mesh of five by five rectangular elements is used in all the analyses. The analyses are carried out with arc-length control enhanced with line searches, aiming at a globally convergent algorithm.

The panel analysed, denoted by WP1, has dimensions  $5000 \times 2800 \times 150 \text{ mm}^3$ . The panel is simply supported in the four edges. Figure 4.2 and Figure 4.3 illustrate the behaviour of the panel at peak load and ultimate load. The development of typical failure lines, similar to the yield line theory of slabs is clearly visible in Figure 4.3. At peak, the failure lines are not fully formed and cannot be obtained.

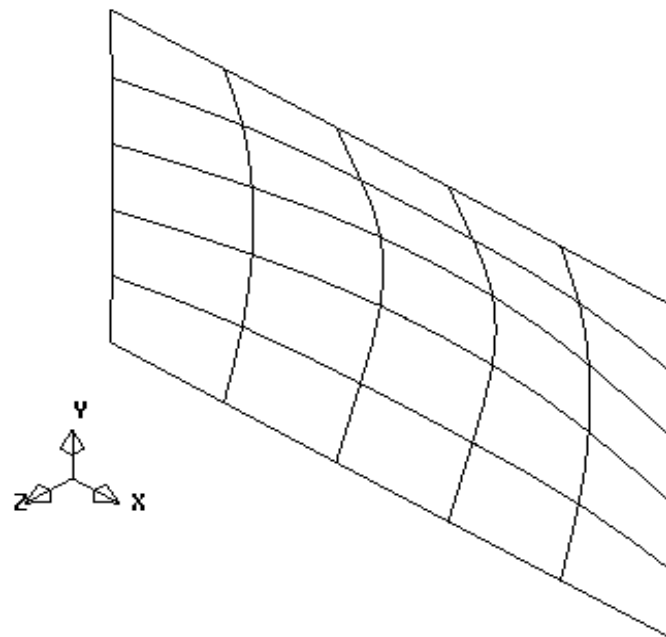


(a)

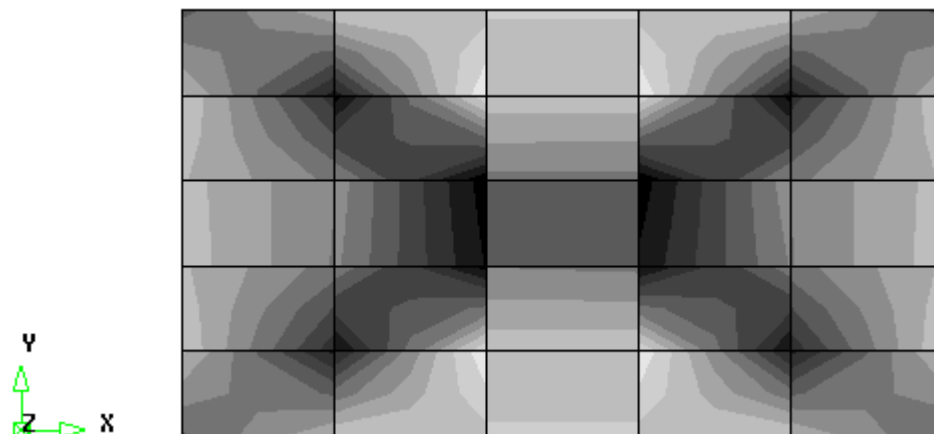


(b)

Figure 4.2 - Panel WP1. Results of the analysis at a pressure  $p$  equal to 9.45 kPa (peak): (a) total deformed mesh; (b) maximum principal strain



(a)



(b)

Figure 4.3 - Panel WP1. Results of the analysis at a pressure  $p$  equal to 8.20 kPa (87% post-peak): (a) total deformed mesh; (b) maximum principal strain

The calculated load-displacement diagram for a displacement  $d$  at the centre of the specimen and the experimental collapse load are given in Figure 4.4. An extremely ductile behaviour is observed and reasonable agreement is found with the experimental value.



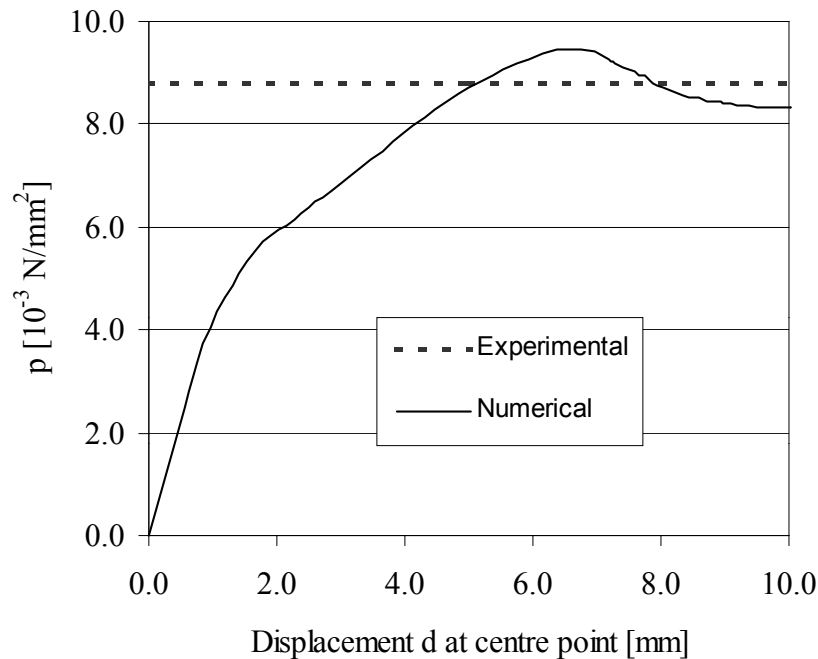


Figure 4.4 - Panels WP1. Load-displacement diagram and experimental failure load (average of three tests).

## 4.2 University of Plymouth solid brick clay masonry panels

The second series considered here consists of a set of solid clay brick masonry panels submitted to out-of-plane loading. The tests were carried out by Chong *et al* (1995) and are denoted by SB. The elastic and inelastic properties of the composite material are given in Table 4.3 and Table 4.4. Only tensile inelastic behaviour is considered and, following Chong *et al* (1995), isotropic material properties are assumed (this can be accepted because the panel is built with solid brick masonry). The adopted Poisson's ratio is 0.2 and the Young's modulus has been obtained by inverse fitting. Please note that, in order to remove the effect of the  $z$ -axis, orthotropic material data must be introduced with the following approximately zero values were assumed:  $E_z = 1.d-3$ ,  $\nu_{xz} = 1.d-7$  and  $\nu_{yz} = 1.d-7$ . The uniaxial tensile strengths and the fracture energies along the material axes  $g_{fx}$  and  $g_{fy}$  are then calculated so that the correct flexural strength is obtained in a pure bending test, see Lourenço (2000). The parameter  $\alpha$  has been obtained by inverse fitting.

Table 4.3 - Elastic material properties for University of Plymouth panels

$E_x$	$E_y$	$E_z$	$\nu_{xy}$	$\nu_{yz}$	$\nu_{xz}$	$G_{xy}$	$G_{yz}$	$G_{xz}$
14000	14000	1.d-3	0.2	1.d-7	1.d-7	5833	5833	5833
$N/mm^2$	$N/mm^2$	$N/mm^2$	-	-	-	$N/mm^2$	$N/mm^2$	$N/mm^2$

Table 4.4 - Inelastic material properties for University of Plymouth panels

Tension regime				
$f_{tx}$	$f_{ty}$	$\alpha$	$g_{fx}$	$g_{fy}$
1.37	0.58	1.1	0.00589	0.00071
$N/mm^2$	$N/mm^2$	-	$N/mm^2$	$N/mm^2$

The panel analysed here, denoted by SB02, yields an additional assessment of the ability of the composite plasticity model to reproduce the behaviour of a panel with an opening of  $2260 \times 1125 \text{ mm}^2$ . Its dimensions are  $5600 \times 2475 \times 100 \text{ mm}^3$ , built in stretcher bond between two stiff abutments with the vertical edges simply supported (allowance for in-plane displacements was provided) and the top edge free. A fixed support was provided at the base because of practical difficulties in providing a simple support. The opening size and dimensions used in the test were chosen to be representative of those used in practice, see Figure 4.5. The panel was loaded by air-bags until failure with increasing out-of-plane uniform pressure  $p$ . The air pressure and the displacement  $d$  for the middle point of the free edge were monitored during testing.

For the numerical analyses eight-noded degenerated shell elements with two by two in-plane Gauss integration and seven-point through thickness integration are used. Due to the symmetry of the structure only one half or one quarter of the panel needs to be modelled. Nevertheless, for easiness in the post-processing and to obtain more clear pictures, the full structure is modelled. The analysis is carried out with arc-length control enhanced with line searches, aiming at a globally convergent algorithm.

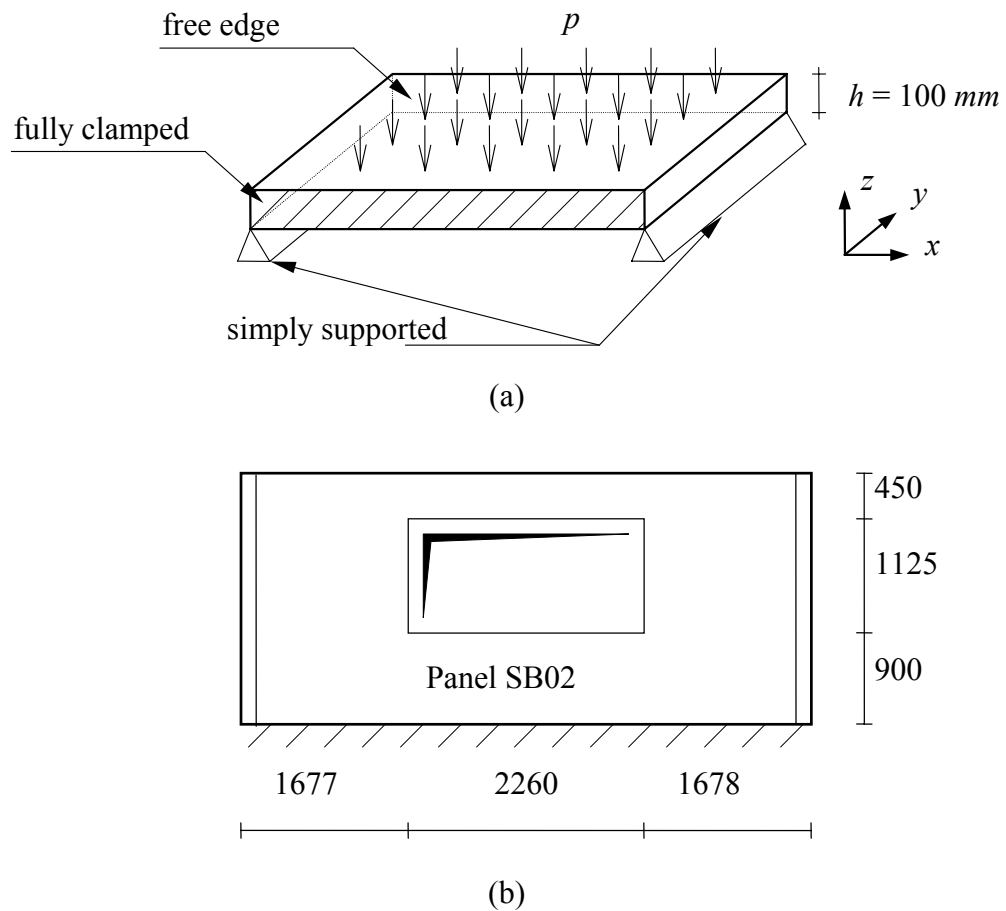
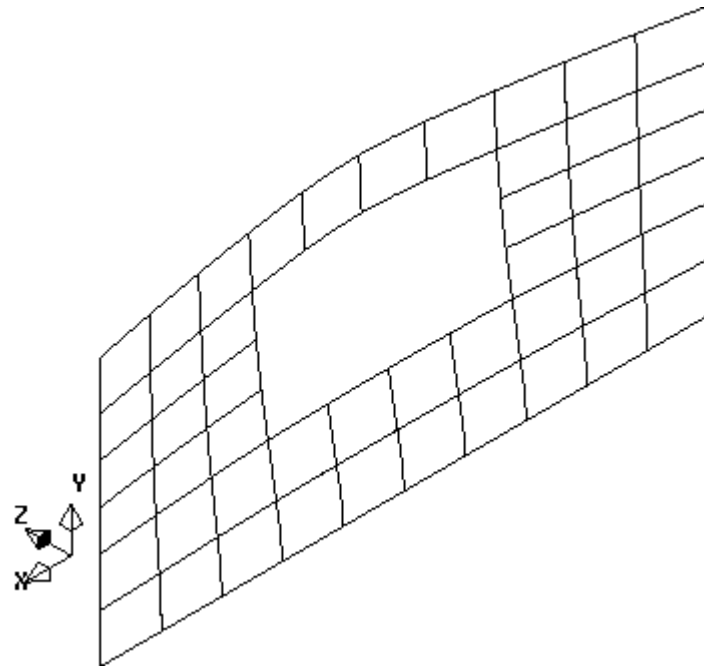
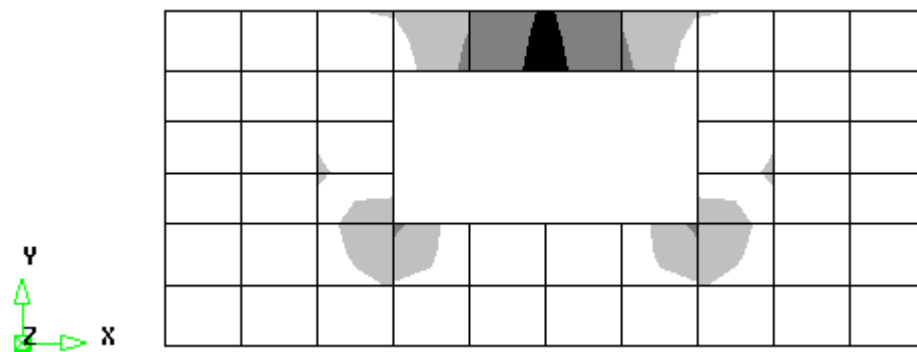


Figure 4.5 - Panels SB02, Chong *et al* (1995): (a) out-of-plane loading;  
(b) geometry, with dimensions in [mm].

Figure 4.6 and Figure 4.7 illustrate the behaviour of the panel at peak load and ultimate load. Cracking is initiated in the middle of the lintel above the central opening, with propagation to the bottom corners. These results are in agreement with the failure mode found experimentally, see Figure 4.8.

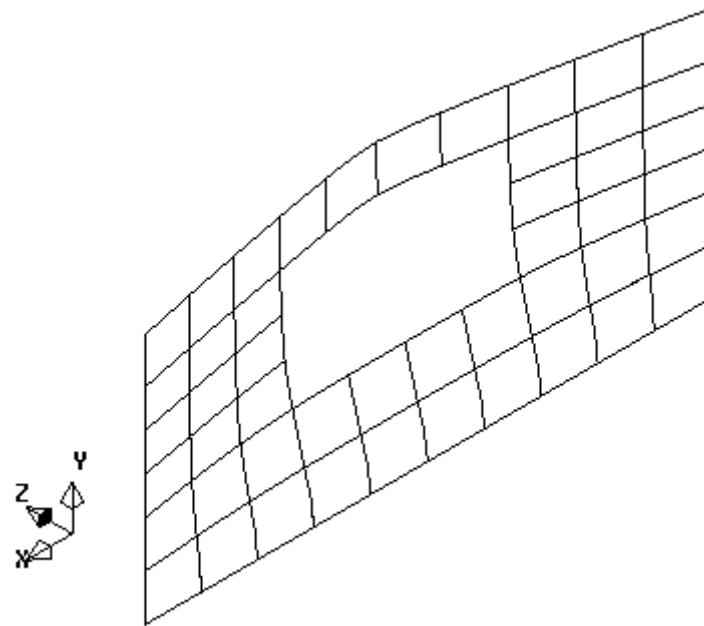


(a)

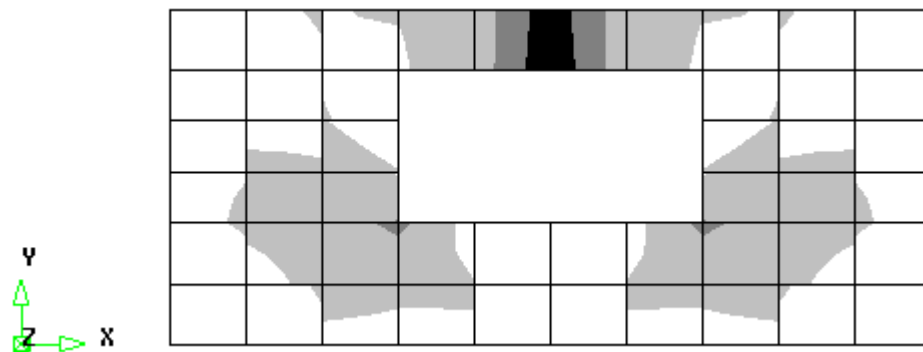


(b)

Figure 4.6 - Panel SB02. Results of the analysis at a pressure  $p$  equal to 2.15 kPa (peak): (a) total deformed mesh; (b) maximum principal strain



(a)



(b)

Figure 4.7 - Panel SB02. Results of the analysis at a pressure  $p$  equal to 1.56 kPa (ultimate): (a) total deformed mesh; (b) maximum principal strain

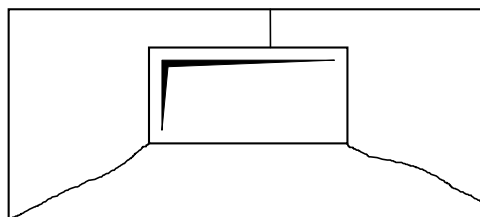


Figure 4.8 – Experimental failure mode.

The calculated and numerical load-displacement diagrams for the displacement  $d$  at the middle of the top (free) edge of the specimen are given in Figure 4.9. Reasonable agreement is found.

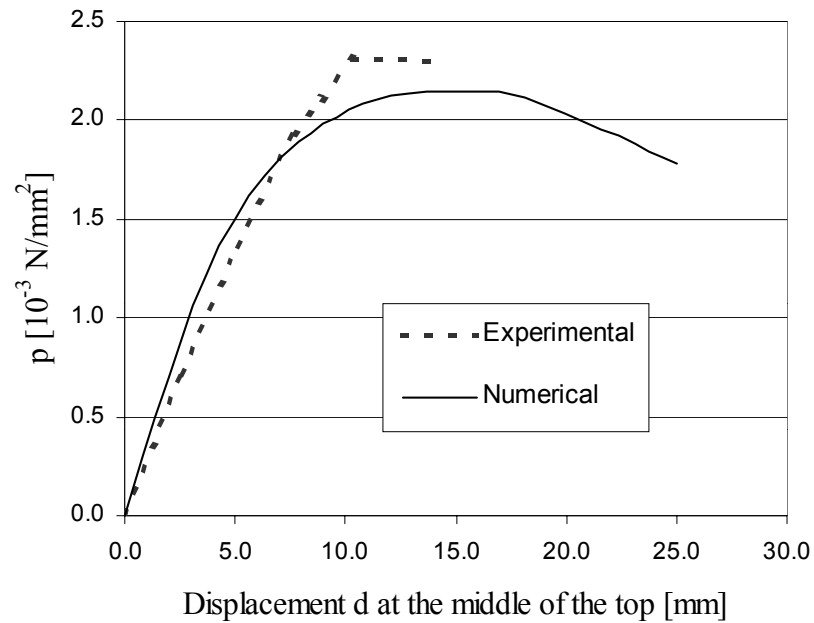


Figure 4.9 - Panel SB02. Load-displacement diagrams.



## 5 Conclusions

An anisotropic model for masonry plates and shells was implemented and assessed. The model can be utilized both with shell elements and three-dimensional elements.

The numerical calculations were compared with results available in the literature for tests in masonry panels subjected to out-of-plane loading. Reasonable agreement was found in all cases. The tests were made of different masonry types and were subjected to different loading conditions. It is, therefore, possible to conclude that the proposed model is of general application for modeling masonry plates and shells.

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# ANNEX A



```

c
c TESTING UNIAXIAL TENSION
c
PROGRAM TEST
INTEGER      nstr, NUV, NUS, ITER
parameter    ( nstr = 4, nuv = 14, nus = 2)
DOUBLE PRECISION DEPS(nstr), se(nstr,nstr), USRVAL(NUV),
$             USRSTA(NUS), tsig(nstr), yt(nstr*nstr),
$             youn(2)
c
c iter = 0
c
c     deps(1) = 2.d-5
c     deps(2) = -4.d-6
c     deps(3) = 0.
c     deps(4) = 0.
c
c Exx = 10000.; Eyy = 5000.; Ezz = 1.d-3
c nu_xy = 0.2; nu_yz = 1.d-7; nu_xz = 1.d-7
c Gxy = 3000.
c se(1,1) = 10200.
c se(1,2) = 1020.
c se(1,3) = 1.224d-10
c se(1,4) = 0.
c se(2,1) = se(1,2)
c se(2,2) = 5102.
c se(2,3) = 1.122d-10
c se(2,4) = 0.
c se(3,1) = se(1,3)
c se(3,2) = se(2,3)
c se(3,3) = 1.d-3
c se(3,4) = 0.
c se(4,1) = se(1,4)
c se(4,2) = se(2,4)
c se(4,3) = se(3,4)
c se(4,4) = 3.d3
c
c tsig(1) = 1.d0
c tsig(2) = 8.d-6
c tsig(3) = 1.d-14
c tsig(4) = 0.
c
c     usrval(1) = 1.0
c     usrval(2) = 0.0002
c     usrval(3) = 0.5
c     usrval(4) = 0.00006
c     usrval(5) = 1.0
c     usrval(6) = 1.0
c     usrval(7) = 1.87d3
c     usrval(8) = 5.0d3
c     usrval(9) = 7.61d3
c     usrval(10) = 10.d3
c     usrval(11) = -1.05
c     usrval(12) = 1.2
c     usrval(13) = 0.0008
c     usrval(14) = 1.
c
c     youn(1) = 10000.
c     youn(2) = 5000.
c
c call USRMAT( DEPS, nstr, 1, 1, se, ITER, USRVAL,
$             NUV, USRSTA, NUS, tsig, yt, youn )
c
c call privec( tsig, nstr, 'TSIG ' )
c call primat( se, nstr, nstr, 'YT ' )
c
c     Quatro iterações para obter:
c
c     tsig(1) = 8.370e-01
c     tsig(2) = -3.629e-02
c     tsig(3) = 7.645e-15
c     tsig(4) = 0.
c
c     se(1,1) = -7.097e03
c     se(1,2) = -7.097e02
c     se(2,2) = 4.929e+03
c     se(2,3) = 9.144e-11
c     se(4,4) = 1.970e+03
c end
c
c SUBROUTINE USRMAT( DEPS, nstr,
$ il, ip, se, ITER, USRVAL,
$ NUV, USRSTA, NUS, tsig, yt, young )
c
c... USER-SUPPLIED MATERIAL MODEL
c... RETURN UPDATED STRESS AND TANGENTIAL STIFFNESS MATRIX
c
c     INTEGER      nstr, NUV, NUS, il, ip, ITER
c     DOUBLE PRECISION DEPS(nstr),
$             se(nstr,nstr), USRVAL(NUV),
$             USRSTA(NUS), tsig(nstr), yt(nstr*nstr),
young(2)
c
c-----
c-----
c
C... ARGUMENTS:
C...     deps In D(nstr) - Strain increment
C...     nstr In I - Dimension of stress
C...     space (must be 4 or 6)
C...     il In I - Number of element
C...     ip In I - Number of integration
C...     point
C...     se In D(nstr,nstr) - Linear elastic stress
C...     strain relation
C...     iter Out I - Number of internal
C...     iterations for stress update
C...     usrval In D(nuv) - Parameters for yield
C...     surface
C...     nuv In I - Number of parameters for
C...     yield surface (equal to 14)
C...     usrsta InOut D(nus) - State variables
C...     nus In I - Number of state
C...     variables (equal to 2)
C...     tsig InOut D(nstr) - Stress at the
C...     beginning of the step / Stress update
C...     yt Out D(nstr,nstr) - Tangent stress strain
C...     relation
C...     youn In D(2) - Young modulus Exx and
C...     Eyy - OUT-OF-PLANE FAILURE IGNORED!!!
C...
C-----
C...
C...     usrval D(14) - Yield surface parameters
C...     (1) : ftx
C...     (2) : Gftx
C...     (3) : fty
C...     (4) : Gfty
C...     (5) : alpha
C...     (6) : alpha sub g
C...     (7) : fmx
C...     (8) : Gfcx
C...     (9) : fmy
C...     (10) : Gfcy
C...     (11) : beta
C...     (12) : gamma
C...     (13) : kappa_peak_compression
C...     (14) : h (length scale)
C...
C-----
C...
C...     usrsta D(2) - State variables
C...     (1) : equivalent plastic strain tension
C...     (2) : equivalent plastic strain compression
C...
C-----
C...
C...     yldpr D(12) - Yield properties
C...     (1) : ftx(kappa)
C...     (2) : fty(kappa)
C...     (3) : alpha
C...     (4) : alpha sub g
C...     (5) : fmx(kappa)
C...     (6) : fmy(kappa)
C...     (7) : beta
C...     (8) : gamma
C...     (9) : d(sigq)/d(epsq)tx
C...     (10) : d(sigq)/d(epsq)ty
C...     (11) : d(sigq)/d(epsq)mx
C...     (12) : d(sigq)/d(epsq)my
C...
C-----
C...
C...     externals
C...     double precision dasum
c
c...     common definition
c...     logical sw
c...     common /SWITCH/ sw(6)
c
c
c...     double precision youn
c...     common /YOUNG/ youn(2)
c
c
c...     local variables
c...     logical lapex
c...     integer mstr, miter
c...     double precision tolyfu
c...     parameter ( mstr = 6, miter = 30, tolyfu = 1.d-5 )
c
c...     double precision yldpr(14), dsig(mstr), depse(mstr),
c...     depsp(mstr), depeq(2), epegyf(2)
c
c...     double precision jacob(10*10), invmat(10,10), dl(2),
c...     fx(10),
c...     $ x(10), a(mstr,mstr), b(mstr,mstr),
c...     $ s(10), gradf(mstr), f, fold, gradnt,
c...     $ tsign(mstr), tsigt(mstr), yf(2),
c...     w(mstr*mstr),
c...     $ hess(10,10), norml, search
c
c

```



```

integer          iiter, ierr, rp(10), nx, iapex, iyld,
ix, jx,
$              i, j
c
double precision delold
c
data rp / 1, 2, 3, 4, 5, 6, 7, 8, 9, 10 /
c
sw(3) = .true.
c
youn(1) = young(1)
youn(2) = young(2)
c
c... Preliminary Checks
c
if ( nstr .ne. 4 .and. nstr.ne. 6 ) THEN
3D' print *, 'SUBROUTINE ONLID VALID FOR PLANE STRAIN AND
print *, 'PLEASE CHANGE SUBROUTINE'
stop
end if
c
if ( nuv .ne. 14 ) THEN
print *, 'WRONG NUMBER OF MATERIAL PARAMETERS <> 14'
print *, 'PLEASE PROVIDE AS INPUT:'
print *,
$ 'ftx Gfx fty Gfy alpha alpha_g fmx Gfcx fmy
Gfcy . beta gamma eps_peak h'
stop
end if
c
if ( nus .ne. 2 ) THEN
print *, 'WRONG NUMBER OF STATE VARIABLES <> 2'
print *, 'PLEASE PROVIDE AS INPUT: 0.0 0.0'
stop
end if
c=====
c... Material Parameters - Definition of the yeldpr vector
c=====
yldpr(1) = usrval(1)
yldpr(2) = usrval(3)
yldpr(3) = usrval(5)
yldpr(4) = usrval(6)
yldpr(5) = usrval(7)
yldpr(6) = usrval(9)
yldpr(7) = usrval(11)
yldpr(8) = usrval(12)
c
iyld = 3
epeqyf(1) = usrsta(1)
epeqyf(2) = usrsta(2)
call rset( 0.d0, depeq, 2 )
c=====
c... set up total test stress
c=====
call rab( se, nstr, nstr, deps, 1, dsig)
call uvpw( tsig, dsig, nstr, tsigt )
if ( sw(3) ) call privec( tsigt, nstr, 'TSIGT' )
c
600 continue
linac = .false.
nx = nstr + 2
call rset( 0.d0, x, nx )
call rmove( tsigt, x, nstr )
call rmove( tsigt, tsign, nstr )
if ( sw(3) ) call privec( x, nx, 'X' )
call ypropx( usrval, epeqyf, depeq, yldpr )
if ( sw(3) ) call privec( yldpr, 12, 'YLDPR' )
c
if ( ( yldpr(5) .lt. 5.d-3 ) .or. ( yldpr(6) .lt. 5.d-3 )
) then
call rset( 0.d0, tsign, nstr )
iyld = 4
iapex = 1
goto 300
end if
c
c=====
c... First value of iyld
c=====
call masyf( yldpr, tsigt, yf, nstr, lapex, usrval, epeqyf
)
if ( yf(1) .le. 1.d-12 .and. yf(2) .le. 1.d-12 ) then
Behaviour remains elastic
call rmove( tsigt, tsig, nstr )
call rmove( se, yt, nstr*nstr )
return
end if
if ( yf(2) .gt. 0.d0 .and. iyld .eq. 3 ) then
iyld = 2
nx = nstr + 1
else if (yf(1) .gt. 0.d0 .and. (iyld .eq. 3 .or. iyld .eq.
2))then
iyld = 1
nx = nstr + 1
else
iyld = 3
nx = nstr + 2
end if
c
iapex = 0
lapex = .false.
call rset( 0.d0, dl, 2 )
c
c=====
c Plastic predictor-corrector
c=====
do 100, iiter = 1, miter
if ( sw(3) ) call priivl( iiter, 'ITER ' )
999 continue
call mafx( fx, x, se, tsigt, nstr, iyld, lapex,
usrval, epeqyf,
$ yldpr)
if ( sw(3) ) call privec( fx, nx, 'fx-ok' )
if ( sw(3) ) call priivl( iyld, 'yld ' )
if ( lapex ) goto 500
if ( iyld .eq. 1 ) then
call jaco12( usrval, epeqyf, jacob, se, nstr, x,
nx, iyld,
$ yldpr )
if ( sw(3) ) call primat( jacob, nx, nx, 'jacob' )
end if
if ( iyld .eq. 2 ) then
call jaco22( usrval, epeqyf, jacob, se, nstr, x,
nx, iyld,
$ yldpr )
if ( sw(3) ) call primat( jacob, nx, nx, 'jacob' )
end if
if ( iyld .eq. 3 ) then
call jaco32( usrval, epeqyf, jacob, se, nstr, x,
nx, iyld,
$ yldpr )
if ( sw(3) ) call primat( jacob, nx, nx, 'jacob' )
end if
call invpqb( jacob, nx, rp, invmat, 1.d-10, ierr )
call rab( jacob, nx, nx, fx, 1, s )
if ( iyld .ne. 1 ) then
call uvmw( x, s, nx, x )
search = 1.d0
else
call lnsea2( fx, x, s, nx, se, tsigt, nstr, iyld,
search,
$ lapex, usrval, epeqyf, yldpr )
end if
c=====
cc... -1
cc... IF THE LINE SEARCH -> 0 THEN ASSUME [J]
SINGULAR.
cc... PERTURB JACOBIAN AND RESTART
c=====
if ( search .lt. 1.d-3 ) then
call invpqb( jacob, nx, rp, invmat, 1.d-10, ierr )
call rabt( jacob, nx, nx, jacob, nx, hess )
if ( sw(3) ) call primat( hess, nx, nx, 'hess' )
c=====
c NORM 1 of HESSIAN
c=====
call rset( 0.d0, w, nx )
norm1 = 0.d0
do 101 ix = 1, nx
do 102 jx = 1, nx
w(ix) = w(ix) + abs( hess( jx, ix ) )
102 continue
101 if ( w(ix) .gt. norm1 ) norm1 = w(ix)
continue
call unimax( invmat, nx )
call uvpws( hess, invmat, nx*nx, norm1 * 5.d-8,
hess )
if ( sw(3) ) call primat( hess, nx, nx, 'hess' )
call invpqb( hess, nx, rp, invmat, 1.d-10, ierr )
call rabt( hess, nx, nx, jacob, nx, invmat )
call rab( invmat, nx, nx, fx, 1, s )
if ( sw(3) ) call primat( invmat, nx, nx, 'new-ja'
)
call lnsea2( fx, x, s, nx, se, tsigt, nstr, iyld,
search,
$ lapex, usrval, epeqyf, yldpr )
end if

```



```
if ( x(nstr+1) .lt. 0.d0 ) x(nstr+1) =0.d0
if ( x(nstr+2) .lt. 0.d0 ) x(nstr+2) =0.d0
c=====
c      Check for convergence
c=====
if ( dasum( nx, fx, 1 ) .le. ( tolyfu * nx ) ) goto
300
call exma( tsign, dl, depeq, x, iyld, nstr )
if( sw(3) ) then
  call prvec( fx, nx, 'F(X)' )
  call prvec( tsign, nstr, 'TSIGN' )
  call prvec( x, nx, 'X' )
  call prival( tolyfu, 'TOLYFU' )
  call prvec( dl, 2, 'dl' )
  call prival( epeqyf(1)+depeq(1), 'EPEQ1' )
  call prival( epeqyf(2)+depeq(2), 'EPEQ2' )
end if
call fxma( tsign, depeq, x, iyld, nstr )
c
100 continue
if ((tsigt(1).lt.yldpr(1)) .or. (tsigt(2).lt.yldpr(2)))
then
  linac = .true.
  call prvec( tsigt, nstr, 'SIGT' )
  call prvec( tsign, nstr, 'SIGN' )
  call exma( tsign, dl, depeq, x, iyld, nstr )
  goto 300
end if
500 continue
c=====
c      Handle apex
c=====
call rset( 0.d0, depeq, 2 )
dl(2) = 0.d0
call ypropx( usrval, epeqyf, depeq, yldpr )
if ( ( yldpr(1) .lt. 1.d-4 ) .and. ( yldpr(2) .lt. 1.d-4 )
) then
  call rset( 0.d0, tsign, 4 )
  iyld = 4
  iapex = 1
  goto 300
end if
iyld = 1
write( *, 4 ) il, ip
call rmove( se, invmat, nstr*nstr )
call invsym( invmat, nstr )
call filma( +1, invmat, nstr )
c=====
c...      Calculate plastic corrector
c=====
call fabra( a, b, nstr, se )
do 400, iiter = 1, miter
  if( sw(3) )call prival( iiter, 'ITER' )
c
call ypropx( usrval, epeqyf, depeq, yldpr )
call rset( 0.d0, w, nstr )
w(1) = yldpr(1)
w(2) = yldpr(2)
call rab( a, nstr, nstr, w, 1, tsign )
call rab( b, nstr, nstr, tsign, 1, w )
call uvpw( tsign, w, nstr, tsign )
c=====
c      Update plastic strains
c=====
call uvmw( tsign, tsign, nstr, w )
call rab( invmat, nstr, nstr, w, 1, depsp )
call uvmw( deps, depsp, nstr, depsp )
c=====
c...      Determine elastic strain increment and
stress_increment
c=====
call uvmw( deps, depsp, nstr, depse )
if( sw(3) ) then
  call prvec( tsign, nstr, 'TSIGN' )
  call prvec( depsp, nstr, 'DEPSP' )
  call prvec( depse, nstr, 'DEPSE' )
  call prival( epeqyf(1)+depeq(1), 'EPEQ1' )
end if
c=====
c...      Calculate new depeq - secant method
c=====
if ( iiter. eq. 1 ) then
  delold = 0.d0
  fold = 0.d0 - (depsp(1)+depsp(2)) / 2.d0 - 0.5d0 *
sqrt(abs((
$      depsp(1) - depsp(2) ) ** 2 + depsp(4) ** 2 /
yldpr(4) ) ) )
  depeq(1) = (depsp(1)+depsp(2)) / 2.d0 - 0.5d0 *
sqrt(abs((
$      depsp(1) - depsp(2) ) ** 2 + depsp(4) ** 2 /
yldpr(4) ) ) )
  else
  f = depeq(1) - (depsp(1)+depsp(2)) / 2.d0 - 0.5d0 *
sqrt(abs((
$      depsp(1) - depsp(2) ) ** 2 + depsp(4) ** 2 /
yldpr(4) ) ) )
  gradnt = ( f - fold ) / ( depeq(1) - delold )
  delold = depeq(1)
  fold = f
  depeq(1) = depeq(1) - ( f / gradnt )
  if( sw(3) ) call prival( f, 'f' )
end if
c
if( sw(3) ) then
  call prival( depeq(1), 'dEPEQ1' )
end if
c=====
c      Check for convergence
c=====
if ((abs(delold-depeq(1))) .le. tolyfu)
$      goto 450
400 continue
linac = .true.
print*, 'Bug in the apex return mapping!'
stop
c
450 continue
dl(1) = depeq(1)
call uvv( depsp, nstr, 1.d0 / dl(1), gradf )
c=====
c      Store apex data
c=====
call ypropx( usrval, epeqyf, depeq, yldpr )
call rset( 0.d0, w, nstr )
w(1) = yldpr(1)
w(2) = yldpr(2)
call rab( a, nstr, nstr, w, 1, tsign )
call rab( b, nstr, nstr, tsign, 1, w )
call uvpw( tsign, w, nstr, tsign )
c=====
c      Update plastic strains
c=====
call uvmw( tsign, tsign, nstr, w )
call rab( invmat, nstr, nstr, w, 1, depsp )
call uvmw( deps, depsp, nstr, depsp )
iapex = 1
iyld = 1
if( sw(3) ) then
  call prival( dl(1), 'dl' )
  call prvec( gradf, nstr, 'GRADF' )
end if
300 continue
if ( iapex .eq. 0 ) call exma( tsign, dl, depeq, x, iyld,
nstr )
c=====
c      Check if another yield surface becomes active
c=====
if ( iyld .ne. 3 ) then
  call ypropx( usrval, epeqyf, depeq, yldpr )
  call masyf( yldpr, tsign, yf, nstr, iapex, usrval,
epeqyf )
c
if ( iyld .eq. 2 .and. yf(1) .gt. 0.d0 ) then
  print*, "MOVE FROM COMPRESSION TO TENSION"
  goto 600
end if
if ( iyld .eq. 1 .and. yf(2) .gt. 0.d0 ) then
  print*, "MOVE FROM TENSION TO CORNER"
  goto 600
end if
end if
c=====
c      Check if the wrong side of the yield surface was selected
c=====
if ( ( iyld .eq. 1 ) .and. ( iapex .eq. 0 ) )
$      call findap( tsigt, nstr, x, yldpr, iapex, usrval,
epeqyf )
c...      update plastic strains
call uvmw( tsign, tsign, nstr, w )
call rmove( se, invmat, nstr*nstr )
call invsym( invmat, nstr )
```



```

call filma( +1, invmat, nstr )
call rab( invmat, nstr, nstr, w, 1, depsp )
c=====
c      determine elastic strain_increment and stress_increment
c=====
call uvmw( deps, depsp, nstr, depse )
c
call uvmw( tsign, tsig, nstr, dsig )
c=====
c...   Show yield values - Only for debug
c=====
      if ( sw(3) ) then
call ypropx( usrval, epeqyf, depeq, yldpr )
call PRIVEC( YLDPR, 8, 'YLDPR' )
      end if
c=====
c..    Calculate equivalent plastic strains and update
stresses
c=====
usrsta(1) = epeqyf(1) + depeq(1)
usrsta(2) = epeqyf(2) + depeq(2)
call rmove( tsign, tsig, nstr )
c print*, "yldpr", epeqyf(1), YLDPR(5)/YLDPR(6)
c print*, "yldpr", epeqyf(2), YLDPR(1)/YLDPR(2)
c
      if ( sw(3) ) write( *, 5 ) il, ip, iter
c-----
      if ( tsigt(4) * tsign(4) .lt. 0.d0 ) then
ip      print*, "Change of sign in shear stress. El: Ip:", il,
      print*, "sigt: sign:", tsigt(4), tsign(4)
      end if
c-----
      if ( linac ) then
      if ( iyld .eq. 1 ) write( *, 1 ) il, ip, fx(nstr+1)
      if ( iyld .eq. 2 ) write( *, 2 ) il, ip, fx(nstr+1)
      if ( iyld .eq. 3 ) write( *, 3 ) il, ip, fx(nstr+1),
      $      fx(nstr+2)
      end if
c=====
c      Calculate consistent tangent operator
c=====
      if ( iapex .eq. 0 ) then
do 80 j = 1, nstr
do 80 i = 1, nstr
      yt( i+(j-1)*nstr ) = jacob(i+(j-1)*nx )
80 continue
      else
call drapex( yt, se, nstr, yldpr, tsign, depsp )
c      call rset( 0.d0, yt, nstr*nstr )
c      yt(1) = 10.d0
c      yt(nstr+2) = 10.d0
c      yt(2*nstr+3) = 10.d0
      end if
c
      if ( sw(3) ) call primat( yt, nstr, nstr, 'YT' )
c
1 FORMAT ( ' ELEMENT:', I4, ' IP:', I4,
$      ' IS INACCURATE - FTENS=', E11.3, ' TENSION
REGIME' )
2 FORMAT ( ' ELEMENT:', I4, ' IP:', I4,
$      ' IS INACCURATE - FCOMP=', E11.3,
$      ' COMPRESSION REGIME' )
3 FORMAT ( ' ELEMENT:', I4, ' IP:', I4,
$      ' IS INACCURATE - FTENS=', E11.3, ' FCOMP=',
E11.3,
$      ' CORNER REGIME' )
4 FORMAT ( ' ELEMENT:', I4, ' IP:', I4, ' IS IN THE
APEX' )
5 FORMAT ( ' ELEMENT:', I4, ' IP:', I4, ' NO. OF INT.
ITER: ', I2)
      end
c
c      SUBROUTINE YPROPX( usrval, epeqyf, DEPEQ, YLDPR )
C.....
C... PURPOSE:
C... UPDATING OF THE YIELD-CRITERION PROPERTIES
C... TAKING INTO ACCOUNT: -HARDENING
C...
C.....
C.....
C
DOUBLE PRECISION usrval(*), epeqyf(*), DEPEQ(*), YLDPR(*)
LOGICAL SW
COMMON /SWITCH/ SW(6)
C
C      DOUBLE PRECISION h, xtop, HCT, HCS, HFT, KAPPA, DKAPPA,
sigeq
c
h = usrval(14)
xtop = usrval(13)
c
kappa = epeqyf(1)
dkappa = depeq(1)
c
call fnsigx( 3, usrval(1), 'HARVLX', h, xtop,
usrval(2), kappa,
$      dkappa, sigeq, hct, hcs )
      yldpr(1) = sigeq
      yldpr(9) = hct
c
call fnsigx( 3, usrval(3), 'HARVLY', h, xtop,
usrval(4), kappa,
$      dkappa, sigeq, hct, hcs )
      yldpr(2) = sigeq
      yldpr(10) = hct
c
kappa = epeqyf(2)
dkappa = depeq(2)
c
call fnsigx( 14, usrval(7), 'CMPVLX', h, xtop,
usrval(8), kappa,
$      dkappa, sigeq, hct, hcs )
      yldpr(5) = sigeq
      yldpr(11) = hct
c
call fnsigx( 14, usrval(9), 'CMPVLY', h, xtop,
usrval(10), kappa,
$      dkappa, sigeq, hct, hcs )
      yldpr(6) = sigeq
      yldpr(12) = hct
c
IF ( SW(3) ) CALL PRIVEC( YLDPR, 12, 'YLDPR' )
c
RETURN
END
c
c      SUBROUTINE FNSIGX( IHARFN, sigq0, NAME, h, xtop, gf,
KAPPA,
$      DKAPPA, SIGEQ, HCT, HCS )
C.....
C.....
C.....
C... common definition
logical sw
common /SWITCH/ sw(6)
c
double precision youn
common /YOUNG/ youn(2)
c
INTEGER EXPONE, CURVE3
PARAMETER ( EXPONE=3, CURVE3=14 )
c
INTEGER IHARFN
CHARACTER*6 NAME
DOUBLE PRECISION xmed, sigq0, h, xtop, gf, KAPPA, DKAPPA,
SIGEQ,
$      HCT,HCS, hmax, xult, xval, xmdmax, young
c
INTEGER NHARVL, IDUM
c
C... USER-SUPPLIED EQUIVALENT STRESS
C... DETERMINE INITIAL EQUIVALENT STRESS
c
IF ( IHARFN .EQ. EXPONE ) THEN
if ( name .eq. 'HARVLX' ) then
young = youn(1)
end if
if ( name .eq. 'HARVLY' ) then
young = youn(2)
end if
c
HMAX = GF * YOUNG / ( SIGQ0 * SIGQ0 )
c
IF ( H .GT. HMAX ) THEN
SIGQ0 = SQRT( GF * YOUNG / H )
END IF
XULT = GF / ( H * SIGQ0 )
c
XVAL = KAPPA + DKAPPA
CALL FNCEXP( SIGQ0, XULT, XVAL, SIGEQ, HCT )
sigeq = max( sigeq, 1.d-03*sigq0 )
c
else if ( iharfn .eq. curve3 ) then
if ( name .eq. 'CMPVLX' ) then
young = youn(1)

```



```

else
  young = youn(2)
end if
c
xmed = 75.d0 / 67.d0 * gf / ( h * sigg0 ) + xtop
xmdmax = sigg0 / young + xtop
if ( xmed .lt. xmdmax ) then
  sigg0 = sqrt( 75.d0 / 67.d0 * gf * young / h )
  xmed = xmdmax
end if
c
call fncr3( sigg0, xtop, xmed, kappa+dkappa, sigeg,
hct )
sigeg = max( sigeg, 1.d-03*sigg0 )
c
ELSE
  print*, 'Hardening / softening law not yet
implemented'
  stop
END IF
c
RETURN
END
c
c
subroutine fncr3( fval0, xtop, xmed, x, fval, grad )
c
c.....
c Return yield value and df/dk for parabolic + exponential
curve
c.....
c
double precision fval0, xtop, xmed, x, fval, grad
double precision fval1, fval2, fval3, fval4, m
c
fval1 = 0.3333333d0 * fval0
fval2 = fval0
fval3 = 0.5d0 * fval0
fval4 = 0.1d0 * fval0
c
c... Special case of kappa equal to zero : grad = +inf
if ( x .eq. 0.D0 ) then
  fval = fval1
  grad = 1.d20
  return
end if
c
if ( x .le. xmed ) then
  if ( x .le. xtop ) then
    m = sqrt( abs( ( 2.D0 * x / xtop - x * x /
$ ( xtop * xtop ) ) ) )
    fval = fval1 + ( fval2 - fval1 ) * m
    grad = ( fval2 - fval1 ) * ( 2.D0 / xtop - 2.D0 *
$ x / ( xtop * xtop ) ) / 2.D0 / m
  else
    fval = ( fval3 - fval2 ) / ( xmed - xtop ) ** 2 *
$ ( x - xtop ) ** 2 + fval2
    grad = 2.D0 * ( fval3 - fval2 ) / ( xmed - xtop )
$ ** 2 * ( x - xtop )
  end if
else
  m = 2.D0 * ( fval3 - fval2 ) / ( xmed - xtop )
  fval = fval3 + ( fval4 - fval3 ) * ( 1.d0 -
$ exp( m / ( fval3 - fval4 ) * ( x - xmed ) ) )
  grad = m * exp( m / ( fval3 - fval4 ) * ( x - xmed ) )
end if
c
return
c
end
c
c
c
subroutine masyf( yldpr, sig, yf, nstr, lapex, usrval,
$ epegyf )
c
c.....
c... purpose:
c... to determine the function value of the masonry
c... yield criterion
c...
c... input:
c... arg.list: yldpr -yieldproperties
c... sig -stress values
c... nstr -dimension of stress space
c...
c... output:
c... arg.list: yf -function value of yield criteria.
c... (1) -orthotropic rankine
c... (2) -hill type
c.....
c.....
c
c... arguments
integer nstr
double precision sig(*), yf(*), yldpr(*), usrval(*),
epegyf(*)
c
c... common definition
logical sw
common /switch/ sw(6)
c
c... local variables
integer mstr
parameter ( mstr = 6 )
double precision p(mstr,mstr) , tri(1) , w(mstr),
. eta(mstr),
. pi(mstr), uv
c
c... orthotropic rankine
c
c T (1/2) T
c... yf = ( 0.5 <eta> [p] <eta> ) + 1/2 <pi> <eta>
c
call fptraor( p, nstr, yldpr(3) )
call rset( 0.d0, pi, nstr )
pi(1) = 1.d0
pi(2) = 1.d0
call feta( sig, yldpr, eta, nstr )
c if( sw(4) ) call privec( ETA, nstr, 'ETA ' )
c if( sw(4) ) call primat( P, nstr, nstr, 'P ' )
call ratba( eta, nstr, 1, p, tri(1), w, 1 )
yf(1) = sqrt( 0.5d0 * tri(1) ) + 0.5d0 * uv( pi, eta,
nstr )
c
c... hill type
c T 1/2
c... yf = ( 0.5 <sig> [p] <sig> ) - sigma
c
call fpma( p, nstr, yldpr )
c if( sw(4) ) call primat( p, nstr, nstr, 'P ' )
call ratba( sig, nstr, 1, p, tri(1), w, 1 )
yf(2) = sqrt( 0.5d0 * tri(1) ) - sqrt( yldpr(5)*yldpr(6) )
)
c
end
c
subroutine mafx( fx, x, se, sigt, nstr, iyld, lapex,
usrval,
$ epegyf, yldpr )
c
c.....
c...
c... PURPOSE:
c... To setup and fill the vector of unknowns for the
return mapping
c... of the composite yield surface for masonry
c...
c... OUTPUT:
c... ARG.LIST: f(x) -vector of unknowns
c...
c.....
c
c... common definition
logical sw
common /switch/ sw(6)
c
double precision rpla, uv, depeq(2), usrval(*), epegyf(*)
logical lapex
c
integer mstr, nstr
parameter ( mstr = 6 )
c
double precision se(nstr,*), yldpr(*), sigt(*), fx(*),
x(*)
c
integer ix, iyld
c
double precision sig(mstr), gradg(mstr), gradfc(mstr),
$ w(mstr*mstr), yf(2), psig(mstr),
$ p(mstr,mstr), pc(mstr,mstr)
c
double precision dl(2)
c
ix = nstr + 1
c=====
c Update softening parameters
c=====
call exma( sig, dl, depeq, x, iyld, nstr )
call ypropx( usrval, epegyf, depeq, yldpr )
c=====
c Calculate gradients

```



```

C=====
-----
      call fpraor( p, nstr, yldpr(4) )
      call ragror( p, sig, gradg, yldpr, nstr, lapex )
      if ( lapex ) return
      call fpma( pc, nstr, yldpr )
      call rab( pc, nstr, nstr, sig, 1, psig )
      call uvs( psig, nstr, 0.5d0 / sqrt( 0.5d0 *
$         abs( uv( sig, psig, nstr ) ) ), gradfc )
c
c      IF( SW(3) )CALL PRIVEC( GRADFC, NSTR, 'GRADFC' )
c      IF( SW(3) )CALL PRIVEC( GRADE, NSTR, 'GRADE' )
c
c...      Fill equations relative to sigma unknowns
c...      <FX> = <SIG> - <SIGT> + dl(1) [D] <GRADG> + dl(2) [D]
<GRADFC>
      call uvmw( sig, sigt, nstr, fx )
      call invsym( se, nstr )
      call filma( +1, se, nstr )
      call rab( se, nstr, nstr, fx, 1, w )
      call invsym( se, nstr )
      call filma( +1, se, nstr )
      call uvpws( w, gradg, nstr, dl(1), fx )
      call uvpws( fx, gradfc, nstr, dl(2), fx )
c...
      call masyf( yldpr, sig, yf, nstr, lapex, usrval, epeqyf )
      if ( iyld .eq. 1 .or. iyld. eq. 3 ) then
c
c...      Tension
c...      Fill equation relative to scalar (dl(1))
c...      FX = F
c...      fx(ix) = yf(1)
      ix = ix + 1
      end if
      if ( iyld .eq. 2 .or. iyld. eq. 3 ) then
c
c...      Compression
c...      Fill equations relative to scalar unknown (dl(2))
c...      FX = F
c...      fx(ix) = yf(2)
      ix = ix + 1
      end if
c
c      end
c
c      subroutine exma( sig, dl, depeq, x, iyld, nstr )
c
c.....
c...      arguments
      integer          iyld, nstr, i
      double precision sig(*), x(*), dl(*), depeq(*)
c
      call rmove( x, sig, nstr )
      do 10 i = 1, nstr
        sig(i) = x(i)
      10 continue
      if ( iyld .eq. 1 ) then
        dl(1) = x(nstr+1)
        depeq(1) = x(nstr+1)
        depeq(2) = 0.d0
        dl(2) = 0.d0
      else if ( iyld .eq. 2 ) then
        dl(1) = 0.d0
        depeq(1) = 0.d0
        dl(2) = x(nstr+1)
        depeq(2) = x(nstr+1)
      else
        dl(1) = x(nstr+1)
        depeq(1) = x(nstr+1)
        depeq(2) = x(nstr+2)
        dl(2) = x(nstr+2)
      end if
c
c      end
c
c      subroutine fpraor( p, nstr, tau )
c
c.....
c...
      C... purpose:
      C... to setup and fill the projection-matrix on the
      "orthotropic" rankine
      C... yield condition
      C...
      C... input:
      C... arg.list:
      C...
      C... output:
      C... arg.list: p -projection-matrix
      C...
      C.....
      C
      C... arguments
      integer          nstr
      double precision p(nstr,*), tau
      C
      C... local variables
      integer          i
      double precision r1
      C
      call rset( 0.0d0 , p , nstr*nstr )
      C
      r1 = 0.5D0
      C
      p(1,1) = r1
      p(1,2) = - r1
      p(2,1) = - r1
      p(2,2) = r1
      do 10 i = 4, 4
        p(i,i) = 2.0d0 * tau
      10 continue
      C
      end
      C
      C      subroutine ragror( p, sig, gradf, yldpr, nstr, lapex )
      C
      C.....
      C... purpose:
      C... to determine gradient vector < gradf > for the
      C... "orthotropic" rankine yield criterion
      C...
      C... input:
      C... arg.list: sig -stress values
      C... nstr -dimension of stress space
      C...
      C... /SWITCH/: sw -logical flags for test output.
      C...
      C... output:
      C... arg.list: gradf -gradient vector.
      C...
      C... workspace:
      C... local: grad -gradient vector
      C... p(6,6) -projection matrix
      C... r -constant
      C... tri -triple product <SIG>[P]<SIG>
      C.....
      C...
      C... externals
      double precision uv
      C
      C... arguments
      logical          lapex
      integer          nstr
      double precision sig(*), gradf(*), yldpr(*), p(*)
      C
      C... common definition
      logical sw
      common /SWITCH/ sw(6)
      C
      C... local variables
      integer          mstr
      parameter ( mstr = 6 )
      double precision grad(mstr), r, tri, pi(mstr), eta(mstr)
      C
      lapex = .false.
      call feta( sig, yldpr, eta, nstr )
      call rset( 0.d0 , pi, nstr )
      pi(1) = 1.d0
      pi(2) = 1.d0
      C
      C... <GRAD> = [P] * <ETA>
      call rab( p, nstr, nstr, eta, 1, grad )
      C
      C... TRI = <eta> * [P] * <eta>
      tri = uv( eta, grad, nstr )
      C
      if ( abs( tri ) .lt. 1.d-14 ) then
        if ( sw(3) ) call prival( tri, 'TRI' )
        lapex = .true.
        return
      end if

```





```

c
r = sqrt( 2.0d0 / abs(tri) ) / 2.d0
c
c...      [P] <eta>
c...      <GRADE> = ----- + 1/2 <pi>
c...      2 sqrt {1/2 <eta> [P] <eta> }
c
call uvv( grad, nstr, r, grad )
call uvvps( grad, pi, nstr, 0.5D0, gradf )
c
end
c
c
subroutine fpma( p, nstr, yldpr )
C.....
C...
C... purpose:
C... to setup and fill the projection-matrix of the
compressive part
C... of the masonry yield criteria
C...
C... output:
C... arg.list: p -projection-matrix
C... programmed by P.B.Lourenco
C.....
C... arguments
integer nstr, i
double precision p(nstr,*), yldpr(*)
c
call rset( 0.0d0 , p , nstr*nstr )
c
p(1,1) = 2.d0 * yldpr(6) / yldpr(5)
p(1,2) = yldpr(7)
p(2,1) = yldpr(7)
p(2,2) = 2.d0 * yldpr(5) / yldpr(6)
do 10 i = 4, 4
p(i,i) = 2.d0 * yldpr(8)
10 continue
c
end
c
c
subroutine feta( sig, yldpr, eta, nstr )
C.....
C...
C... purpose:
C... to calculate the stress in the reduced space for the
orthotropic
C... rankine yield criterion
C...
C... input:
C... arg.list: sig -stresses in the original space
c... yldpr -yield properties
C...
C... output:
C... arg.list: eta -stresses in the reduced space
C...
C.....
C... arguments
integer nstr
double precision sig(*), yldpr(*), eta(nstr)
c
call rmove( sig, eta, nstr )
c
eta(1) = eta(1) - yldpr(1)
eta(2) = eta(2) - yldpr(2)
c
end
c=====
c This subroutine calculates the Jacobian for Rankine type
yield criterion
c=====
c
subroutine jacol2( usrval, epeqyf, a, stiff, nstr, x, nx,
iyl,
$ yldpr )
c
C.....
C...
C... purpose:
C... to calculate the Jacobian for Rankine type yield
criterion
C...
C... input:
C... arg.list: stiff -elastic stiffness matrix
C... x -vector of unknowns
C...
C... nx -number of unknowns
C...
C... output:
C... arg.list: a -Jacobian
C...
C.....
C... arguments
integer mstr
parameter ( mstr=6 )
c
C... arguments
integer nstr, nx, iyl
c
double precision yldpr(*), stiff(nstr,*), x(*), a(nx,*),
$ usrval (*), epeqyf(*)
c
C... common definition
logical sw
common /switch/ sw(6)
c
C... local variables
double precision dl(2), sig(mstr)
double precision p(mstr,mstr), depeq(2), eta(mstr),
peta(mstr),
$ d2gds2(mstr,mstr), gradf(mstr),
w(10*10),
$ dnedk(mstr), d2gdk(mstr), dfdk, tri,
gradg(mstr)
integer i, j, k
logical ldum
c
call rset( 0.d0, dnedk, nstr )
call rset( 0.d0, a, 5*5 )
c
C... Update softening parameters
call exma( sig, dl, depeq, x, iyl, nstr )
call yprop( usrval, epeqyf, depeq, yldpr )
c
if ( sw(3) ) then
call prival( yldpr(9), 'HC1' )
call prival( yldpr(10), 'HC2' )
call prival( dl(1), 'DL' )
call privec( sig, nstr, 'SIG' )
end if
c
call fpraor( p, nstr, yldpr(3) )
call feta( sig, yldpr, eta, nstr )
c
C... Calculate gradients
call ragror( p, sig, gradf, yldpr, nstr, ldum )
call fpraor( p, nstr, yldpr(4) )
call ragror( p, sig, gradg, yldpr, nstr, ldum )
call rab( p, nstr, nstr, eta, 1, peta )
tri = sqrt( 0.5d0 * uv( eta, peta, nstr ) )
if ( sw(3) ) then
call privec( gradf, nstr, 'GRADE' )
call privec( gradg, nstr, 'GRADG' )
call privec( eta, nstr, 'ETA' )
end if
c
C... calculate d2g/dsigma2 = d2gds2
call rab( peta, nstr, 1, peta, nstr, w )
call uvv( w, nstr*nstr, -0.25d0 / tri**3, w )
call uvvps( w, p, nstr*nstr, 0.5D0 / tri, d2gds2 )
c
C... calculate d2f/dkdsigma = d2gdk
dnedk(1) = -yldpr(9)
dnedk(2) = -yldpr(10)
call rab( d2gds2, nstr, nstr, dnedk, 1, d2gdk )
if ( sw(3) ) then
call primat( d2gds2, nstr, nstr, 'D2GDS2' )
call primat( stiff, nstr, nstr, 'STIFF' )
call privec( d2gdk, nstr, 'D2GDK' )
end if
c-----
C... Fill matrix [A] with sigma related lines
call invsym( stiff, nstr )
call filma( +1, stiff, nstr )
call uvvps( stiff, d2gds2, nstr*nstr, dl(1), w )
call invsym( stiff, nstr )
call filma( +1, stiff, nstr )
k = 0
do 10 j = 1, nstr
do 10 i = 1, nstr
k = k + 1
a(i,j) = w(k)
10 continue
c
call uvvps( gradg, d2gdk, nstr, dl(1), w )
do 20 i = 1, nstr
a(i,nstr+1) = w(i)
20 continue

```



```

c
c-----
c
c...   Calculate df/dk = dfdk
      dfdk = uv( gradf, dnedk, nstr )
      if ( sw(3) ) then
        call prival( dfdk, 'DFDK' )
      end if
c-----
c...   Fill matrix [A] with yield function related line
      do 50 i = 1, nstr
        a(nstr+1,i) = gradf(i)
      50 continue
c
      a(nstr+1,nstr+1) = dfdk
c
c-----
c
c-----
c   This subroutine fills the vector of unknowns for the
return mapping
c-----
      subroutine fxma( sig, depeq, x, iyld, nstr )
c
c.....
c...
c... purpose:
c...   to fill the vector of unknowns for the return mapping
c...
c... input:
c...   arg.list: sig -stress
c...
c... output:
c...   arg.list: x   -vector of unknowns
c...
c.....
c
c... arguments
      integer      iyld, nstr
      double precision sig(*), x(*), depeq(*)
c
      call rmove( sig, x, nstr )
      if ( iyld .eq. 1 ) then
        x(nstr+1) = depeq(1)
      else if ( iyld .eq. 2 ) then
        x(nstr+1) = depeq(2)
      else
        x(nstr+1) = depeq(1)
        x(nstr+2) = depeq(2)
      end if
c
      end
c-----
c
c   This subroutine calculate the inverse of the matrix A by
a Gauss-Jordan
c algorithm
c-----
      SUBROUTINE INVPGB( A, NA, RP, RA, EPS, IERR )
c
c.....
c...
c... PURPOSE:
c...   MATRIX INVERSION WITH PARTIAL PIVOTING( GAUSS
JORDAN).
c...   THE COMPLETE INVERSE OF A IS PROJECTED ON THE
ORIGINAL MATRIX
c...   [ A ].
c...   < RP > AND < RA > ARE USED FOR ROW INTERCHANGING.
c
c... INPUT:
c...   ARG.LIST: A      -R(NA*NA): MATRIX TO BE INVERTED.
c...             NA     -I(1):   ORDER OF [ A ]
c...             EPS    -R(1):   TOLERANCE FOR PIVOT
c...
c... OUTPUT:
c...   ARG.LIST: A      -R(NA*NA): INVERSE OF [ A ]
c...             IERR   -I(1):   ERROR CODE:
c...                   = 0 INVERSION SUCCESFULL;
c...                   =-1 SINGULARITY
ENCOUNTERED;
c...
c... WORKSPACE:
c...   ARG.LIST: RP     -I(NA):   ROW INDICES OF [ A ]
c...             RA     -R(NA):   WORKSPACE FOR ROW OF [
A ]
c...
c.....
c
c... ARGUMENTS:
      INTEGER      NA, RP(*), IERR
      DOUBLE PRECISION A(NA,*), RA(*), EPS

```

```

C
C...LOCAL
      INTEGER      I, IR, J, K, N
      DOUBLE PRECISION APIV, AR, EPIV, HPIV
C
      IERR = 0
      N = NA
      EPIV = EPS
C
C... INITIALIZE ROW-INDICES:
      DO 10, I = 1, N
        RP(I) = I
      10 CONTINUE
C
C... ROW LOOP:
      DO 200, I = 1, N
C
C... FIND PIVOT
      HPIV = ABS( A(I,I) )
      IR = I
      IF( I .LT. N ) THEN
        DO 50, K = (I+1), N
          APIV = ABS( A(K,I) )
          IF( APIV .GT. HPIV ) THEN
            HPIV = APIV
            IR = K
          END IF
        50 CONTINUE
      END IF
      IF( HPIV .LT. EPIV ) THEN
        IERR = -RP(IR)
        print*, 'INVERSION FAILED'
        RETURN
      END IF
C
      IF( IR .GT. I ) THEN
        INTERCHANGE ROWS:
        DO 60, J = 1, N
          AR = A(I, J)
          A(I, J) = A(IR,J)
          A(IR,J) = AR
        60 CONTINUE
        K = RP(I)
        RP(I) = RP(IR)
        RP(IR) = K
      END IF
C
      APIV = 1.DO/A(I,I)
      CALL UVS( A(1,I), N, APIV, A(1,I) )
      A(I,I) = APIV
C
C... COLUMN LOOP:
      DO 100, J = 1, N
        IF( J .NE. I ) THEN
          APIV = -A(I,J)
          IF( APIV .NE. 0.DO ) THEN
            CALL UVPWS( A(1,J), A(1,I), N, APIV, A(1,J) )
            A(I,J) = APIV * A(I,I)
          ELSE
            A(I,J) = 0.DO
          END IF
        END IF
      100 CONTINUE
C
      200 CONTINUE
C
C... INTERCHANGE COLUMNS:
      DO 250, I = 1, N
C
      DO 210, J = 1, N
        K = RP(J)
        RA(K) = A(I,J)
      210 CONTINUE
C
      DO 220, J = 1, N
        A(I,J) = RA(J)
      220 CONTINUE
C
      250 CONTINUE
      RETURN
      END
      subroutine lnsea2( fx, x, s, nx, se, sigt, nstr, iyld,
dl,
$ lapex, usrval, epegyf, yldpr )
c
c-----
c
c... Line search subroutine for the X update
c... ARGUMENTS:
c...   x(nx) - vector of unknownws
c...   fx(nx) - vector of residuals
c...   s(nx) - x update vector
c...           -1
c...           <s> = [J(x)] . <fx>
c...
c... OUTPUT:
c...   x(nx) - updated vector of unknownws
c...

```



```

c... SEE DENNIS AND SCHNABEL, pp. 147-152
c... PARAMETERS:
c... ALFA controls the reduction on the initial
fx value
c... TOLER overrides the linesearch if fx is
already small
c... T
c... N.B: Toler = 0.5 * <fx> <fx>
c...
C.....
.....
C
c
c... parameters
integer mx, msear
double precision alfa, toler
logical lapex
parameter ( mx = 10, msear = 20, alfa = 1.d-4,
$ toler = 1.d-8 )
c... externals
double precision uv
c
c... common definition
logical sw
common /SWITCH/ sw(6)
c
c... arguments
integer nx, nstr, iyld
double precision fx(*), x(*), s(*), se(nstr,*), sigt(*),
$ usrval(*), epeqyf(*), yldpr(*)
c
c... local variables
integer isear
double precision xnew(mx), fx1(mx), fx2(mx), dl, dprev,
fxx,
$ fx11, fx22, fxfx, mat11, mat12, mat21,
mat22,
$ fval1, fval2, det, a, b
c
if ( nx .gt. mx ) then
print*, "Error in LNSEAR.F - Please correct array
boundaries"
stop
end if
c
dl = 1.d0
fxfx = - uv( fx, fx, nx )
c
call uvmw( x, s, nx, xnew )
if ( sw(3) ) call privec( xnew, nx, 'XTRIAL' )
call mafx( fx1, xnew, se, sigt, nstr, iyld,
$ lapex, usrval, epeqyf, yldpr )
fxx = uv( fx, fx, nx ) / 2.d0
fx11 = uv( fx1, fx1, nx ) / 2.d0
if ( sw(3) ) then
call prival( fx11, 'FX-NEW' )
call prival( fxx + alfa * dl * fxfx, 'FX-INI' )
end if
if ( ( fx11 .lt. 2.d0 * ( fxx + alfa * dl * fxfx ) )
$ .or. ( fx11 .lt. toler ) ) then
call rmove( xnew, x, nx )
return
end if
c... Line search correction - Quadratic backtrack
dprev = dl
dl = -0.5d0 * fxfx / ( fx11 - fxx - fxfx )
call chekd( dprev, dl )
if ( sw(3) ) call prival( dl, 'LNSEAR' )
call uvpws( x, s, nx, -dl, xnew )
c
do 100, isear = 1, msear
c
call mafx( fx2, xnew, se, sigt, nstr, iyld,
$ lapex, usrval, epeqyf, yldpr )
fx22 = uv( fx2, fx2, nx ) / 2.d0
if ( sw(3) ) then
call prival( fx22, 'FX-NEW' )
call prival( fxx + alfa * dl * fxfx, 'FX-INI' )
end if
if ( fx22 .lt. ( fxx + alfa * dl * fxfx ) ) then
call rmove( xnew, x, nx )
return
end if
c... Line search correction - Cubic backtrack
mat11 = 1.d0 / dl / dl
mat12 = -1.d0 / dprev / dl
mat21 = -dprev / dl / dl
mat22 = dl / dprev / dprev
c
fval1 = fx22 - fxx - fxfx * dl
fval2 = fx11 - fxx - fxfx * dprev
c
det = dl - dprev
c
a = ( mat11 * fval1 + mat12 * fval2 ) / det
b = ( mat21 * fval1 + mat22 * fval2 ) / det
c
dprev = dl
dl = ( - b + sqrt( b * b - 3.d0 * a * fxfx ) ) / 3.d0
/ a
call chekd( dprev, dl )
if ( sw(3) ) call prival( dl, 'LNSEAR' )
fx11 = fx22
call uvpws( x, s, nx, -dl, xnew )
c
100 continue
c
if ( sw(3) )
$ print*, "Line search did not converge after 10
iterations"
call rmove( xnew, x, nx )
return
c
end
c
subroutine chekd( dprev, dl )
c
c... PARAMETERS
double precision mintol, maxtol
parameter ( mintol = 0.1d0, maxtol = 0.5d0 )
C
C... ARGUMENTS
double precision dprev, dl
c
dl = min( dl, maxtol * dprev )
dl = max( dl, mintol * dprev )
c
return
c
end
c
Returns the identity matrix
c
subroutine unimax( matrix, l )
c
integer i,l
double precision matrix(l,l)
c
call rset(0.d0, matrix, l*l )
do 10, i = 1, l
10 matrix(i,i) = 1.d0
c
return
c
end
c
subroutine jaco22( usrval, epeqyf, a, stiff, nstr, x, nx,
iyld,
$ yldpr )
C
C.....
C... purpose:
C... to calculate the Jacobian for Hill type yield
criterion
C...
C... input:
C... arg.list: stiff -elastic stiffness matrix
C... x -vector of unknowns
C... nx -number of unknowns
C...
C... output:
C... arg.list: a -Jacobian
C...
C.....
.....
c... externals
double precision uv
c
c... parameters
integer mstr
parameter ( mstr=6 )
c
c... arguments
integer nstr, nx, iyld
c
double precision yldpr(*), stiff(nstr,*), x(*), a(nx,*),
$ usrval (*), epeqyf(*)
c... common definition
logical sw
common /switch/ sw(6)
c
c... local variables
double precision dl(2), sig(mstr)
double precision p(mstr,mstr), depeq(2), gradf(mstr),
w(10*10),
$ dfdk, tri, dpdk(mstr*mstr), psig( mstr),
yield, d2fds2(mstr,mstr), dfdkds(mstr)
integer i, j, k

```



```

C
  call rset( 0.d0, a, nx*nx )
  call rset( 0.d0, dpdk, nstr*nstr )
c
c...   Update softening parameters
  call exma( sig, dl, depeq, x, iyld, nstr )
  call ypropx( usrval, epeqyf, depeq, yldpr )
  call fpma( p, nstr, yldpr )
c
  if ( sw(3) ) then
    call prival( yldpr(11), 'HC1C' )
    call prival( yldpr(12), 'HC2C' )
    call prival( dl(2), 'DL ' )
    call privec( sig, nstr, 'SIG ' )
    call primat( p, nstr, nstr, 'P' )
  end if
c
c...   Calculate gradient
  call rab( p, nstr, nstr, sig, 1, psig )
  yield = sqrt ( 0.5d0 * abs ( uv( sig, psig, nstr ) ) )
  call uvs( psig, nstr, 0.5d0 / yield, gradf )
c
c...   Calculate d[P]/dk
  dpdk(1) = 2.d0 * ( yldpr(12) * yldpr(5) - yldpr(11) *
yldpr(6) )
  $ / yldpr(5) / yldpr(5)
  dpdk(nstr+2) = 2.d0 * ( yldpr(11) * yldpr(6) - yldpr(12)
*
$ yldpr(5) ) / yldpr(6) / yldpr(6)
c
  if ( sw(3) ) call primat( dpdk, nstr, nstr, 'DPDK' )
c
c...   Calculate dfdk = <SIG> d[P]/dk <SIG> / ( 4 yield ) -
d(yield)/dk
  call rab( dpdk, nstr, nstr, sig, 1, w )
  tri = uv( sig, w, nstr )
  dfdk = 0.25d0 * tri / yield - 0.5d0 *
$ ( yldpr(12) * YLDPR(5) + yldpr(11) * YLDPR(6) ) /
$ sqrt( yldpr(5) * yldpr(6) )
  if ( sw(3) ) call prival( dfdk, 'dfdk' )
c
c
c
c...   Calculate d2fds2 = [P] / ( 2 yield ) - [P]<SIG>
<SIG>[P] / ( 4 yield )
  call rab( psig, nstr, 1, psig, nstr, w )
  call uvs( w, nstr*nstr, -0.25d0 / yield ** 3, w )
  call uvpws( w, p, nstr*nstr, 0.5d0 / yield, d2fds2 )
  if ( sw(3) ) call primat( d2fds2, nstr, nstr, 'D2FDS2' )
c
c...   Calculate d2fdkds = d[P]/dk <SIG> / ( 2 yield ) -
c...
3
c...   ( <SIG> d[P]/dk <SIG> ) [P] <SIG>
/ ( 8 yield )
  call rab( dpdk, nstr, nstr, sig, 1, w )
  call uvs( w, nstr*nstr, -0.5d0 / yield, w )
  call uvpws( w, psig, nstr, - tri / 8.d0 / yield ** 3,
dfdkds )
  if ( sw(3) ) call privec( dfdkds, nstr, 'DFDKDS' )
c-----
c...   FILL MATRIX A WITH SIGMA RELATED LINES
  call invsym( stiff, nstr )
  call filma( +1, stiff, nstr )
  call uvpws( stiff, d2fds2, nstr*nstr, dl(2), w )
  call invsym( stiff, nstr )
  call filma( +1, stiff, nstr )
  k = 0
  do 10 j = 1, nstr
    do 10 i = 1, nstr
      k = k + 1
      a(i,j) = w(k)
  10 continue
c
  call uvpws( gradf, dfdkds, nstr, dl(2), w )
  do 20 i = 1, nstr
    a(i,nstr+1) = w(i)
  20 continue
c
c-----
c...   FILL MATRIX A WITH YIELD FUNCTION RELATED LINE
  do 50 i = 1, nstr
    a(nstr+1,i) = gradf(i)
  50 continue
c
  a(nstr+1,nstr+1) = dfdk
c
c-----
  if ( sw(3) ) call primat( a, 5, 5, 'A' )
c
  end
c
  subroutine fabra( a, b, nstr, stiff )
c
C.....
C...
C... purpose:
C...   to setup and fill the return mapping matrices of the
C...   "orthotropic" Rankine type yield condition in the apex
C...
C...   input:
C...   arg.list:
C...
C...   output:
C...   arg.list: a,b -projection-matrices
C...
C...   programmed by P.B.Lourenco
C.....
C
C...   arguments
  integer nstr
  double precision a(nstr,*), b(nstr,*), stiff(nstr,*)
c
  call rset( 0.0d0 , a , nstr*nstr )
  call rset( 0.0d0 , b , nstr*nstr )
c
  call invsym( stiff, nstr )
  call filma( +1, stiff, nstr )
c
  call unimax( a, nstr )
  a(3,1) = - stiff(3,1) / stiff(3,3)
  a(3,2) = - stiff(3,2) / stiff(3,3)
c
  b(3,1) = stiff(3,1) / stiff(3,3)
  b(3,2) = stiff(3,2) / stiff(3,3)
  b(3,3) = 1.d0
  if ( nstr .eq. 6 ) then
    b(5,5) = 1.d0
    b(6,6) = 1.d0
  end if
c
  call invsym( stiff, nstr )
  call filma( +1, stiff, nstr )
c
  end
c
  subroutine findap( tsigt, nstr, x, yldpr, lapex, usrval,
epeqyf )
C
C.....
C...
C... purpose:
C... check if the wrong side of the rankine yield surface
was selected
C...
C.....
C
C...   externals
  double precision uv
c
c...   parameters
  integer mstr
  parameter ( mstr=6 )
c
c...   arguments
  integer nstr
  logical lapex
c
  double precision yldpr(*), tsigt(*), x(*), usrval(*),
epeqyf(*)
c
c...   common definition
  logical sw
  common /SWITCH/ sw(6)
c...   local variables
  double precision dl(2), sig(mstr)
  double precision p(mstr,mstr), depeq(2), eta(MSTR),
pi(mstr),
$ dsig(mstr), pdsig(mstr), rdum, pidsig,
k, yf(2)
c
  call rset( 0.d0, pi, nstr )
  k = 0.d0
c
c...   Update softening parameters
  call exma( sig, dl, depeq, x, 1, nstr )
  call ypropx( usrval, epeqyf, depeq, yldpr )
c
  call uvmw( tsigt, sig, nstr, dsig )
  if ( sw(3) ) call privec( tsigt, nstr, 'tsigt' )
  if ( sw(3) ) call privec( sig, nstr, 'tsign' )
c
  call fpraor( p, nstr, yldpr(3) )
  pi(1) = 1.d0
  pi(2) = 1.d0
  call feta( sig, yldpr, eta, nstr )
  if ( sw(3) ) call privec( eta, nstr, 'ETA ' )
c
  call rab( p, nstr, nstr, dsig, 1, pdsig )
  pidsig = uv( pi, dsig, nstr )
  rdum = uv( dsig, pdsig, nstr ) - 0.5d0 * pidsig * pidsig
  if ( abs( rdum ) .lt. 1.d-8 ) return

```



```

k = ( uv( pi, eta, nstr ) * pidsig -
$ 2.d0 * uv( eta, pdsig, nstr ) ) / rдум
if ( sw(3) ) call prival( k, 'K' )
if ( k .gt. 0.d0 .and. k .lt. 1.d0 ) then
c
c... Other solution might exist
call uvpws( sig, dsig, nstr, k, sig )
call masyf( yldpr, sig, yf, nstr, lapex, usrval,
epeqyf )
IF ( SW(3) ) call prival( yf(1), 'yf-new' )
if ( abs( yf(1) ) .lt. 1.d-6 ) then
c... other solution exists
print*, "Other solution exists for apex
solution..."
end if
end if
c
end

subroutine jaco32( usrval, epeqyf, a, stiff, nstr, x, nx,
iyld,
$ yldpr )
c
c.....
c... purpose:
c... to calculate the Jacobian for the corner of the
c... composite masonry criterion
c...
c... input:
c... arg.list: stiff -elastic stiffness matrix
c... x -vector of unknowns
c... nx -number of unknowns
c...
c... output:
c... arg.list: a -Jacobian
c...
c.....
c
c... externals
double precision uv
c
c... parameters
integer mstr
parameter ( mstr=6 )
c
c... arguments
integer nstr, nx, iyld
c
double precision yldpr(*), stiff(NSTR,*), x(*), a(nx,*),
$ usrval (*), epeqyf(*)
c
c... common definition
logical sw
common /switch/ sw(6)
c
c... local variables
double precision dl(2), sig(mstr)
double precision p(mstr,mstr), depeq(2), eta(mstr),
peta(mstr),
$ d2gds2(mstr,mstr), gradf(mstr),
w(10*10),
$ dnedk(mstr), d2gdk(mstr), dfldk, tri,
$ gradg(mstr), pc(mstr,mstr),
gradfc(mstr), tric,
$ psig(mstr), dfdk, yield,
d2fds2(mstr,mstr),
$ dfdkds(mstr), dpdk(mstr*mstr),
gamma(mstr)
integer i, j, k
logical ldum

call rset( 0.d0, dnedk, nstr )
call rset( 0.d0, a, nx*nx )
call rset( 0.d0, dpdk, nstr*nstr )
c
c... Update softening parameters
call exma( sig, dl, depeq, x, iyld, nstr )
call ypropx( usrval, epeqyf, depeq, yldpr )
c
if ( sw(3) ) then
call prival( yldpr( 9), 'HC1' )
call prival( yldpr(10), 'HC2' )
call prival( yldpr(11), 'HC1C' )
call prival( yldpr(12), 'HC2C' )
call prival( dl(1), 'DL1' )
call prival( dl(2), 'DL2' )
call privec( sig, nstr, 'SIG' )
end if
c
c... Compression data
call fpma( pc, nstr, yldpr )
c
c... Calculate gradient
call rab( pc, nstr, nstr, sig, 1, psig )
yield = sqrt( 0.5d0 * abs( uv( sig, psig, nstr ) ) )

call uvs( psig, nstr, 0.5d0 / yield, gradfc )
c
c... Calculate d[P]/dk
dpdk(1) = 2.d0 * ( yldpr(12) * yldpr(5) - yldpr(11) *
yldpr(6) )
$ / yldpr(5) / yldpr(5)
dpdk(nstr+2) = 2.d0 * ( yldpr(11) * yldpr(6) - yldpr(12)
*
$ yldpr(5) ) / yldpr(6) / yldpr(6)
c
c...
T
c... Calculate dfdk = <SIG> d[P]/dk <SIG> / ( 4 yield ) -
d(yield)/dk
call rab( dpdk, nstr, nstr, sig, 1, w )
tric = uv( sig, w, nstr )
dfdk = 0.25d0 * tric / yield - 0.5d0 *
$ ( yldpr(12) * yldpr(5) + yldpr(11) * yldpr(6) ) /
$ sqrt( yldpr(5) * yldpr(6) )
c
c...
T
3
c... Calculate d2fds2 = [P] / ( 2 yield ) - [P]<SIG>
<SIG>[P] / ( 4 yield )
call rab( psig, nstr, 1, psig, nstr, w )
call uvs( w, nstr*nstr, -0.25d0 / yield ** 3, w )
call uvpws( w, pc, nstr*nstr, 0.5d0 / yield, d2fds2 )
c
c... Calculate d2fdkds = d[P]/dk <SIG> / ( 2 yield ) -
T
3
c... ( <SIG> d[P]/dk <SIG> ) [P] <SIG>
/ ( 8 yield )
call rab( dpdk, nstr, nstr, sig, 1, w )
call uvs( w, nstr*nstr, 0.5d0 / yield, w )
call uvpws( w, psig, nstr, - tric / 8.d0 / yield ** 3,
dfdkds )
c
call uvpws( gradfc, dfdkds, nstr, dl(2), gamma )
c
c
if ( sw(3) ) then
call primat( d2fds2, nstr, nstr, 'D2FDS2' )
call primat( dpdk, nstr, nstr, 'DPDK' )
call privec( dfdkds, nstr, 'DFDKDS' )
call privec( gamma, nstr, 'GAMMA' )
call prival( dfdk, 'DFDK' )
end if
c
c... End compression
c
c... Begin tension
c
call fptraor( p, nstr, yldpr(3) )
call feta( sig, yldpr, eta, nstr )
c
c... Calculate gradients
call ragror( p, sig, gradf, yldpr, nstr, ldum )
call fptraor( p, nstr, yldpr(4) )
call rab( p, nstr, nstr, eta, 1, peta )
tri = sqrt( 0.5d0 * uv( eta, peta, nstr ) )
call ragror( p, sig, gradg, yldpr, nstr, ldum )
if ( sw(3) ) then
call privec( gradf, nstr, 'GRADF' )
call privec( gradg, nstr, 'GRADG' )
call privec( eta, nstr, 'ETA' )
end if
c
c... calculate d2g/dsigma2 = d2gds2
call rab( peta, nstr, 1, peta, nstr, w )
call uvs( w, nstr*nstr, -0.25d0 / tri**3, w )
call uvpws( w, p, nstr*nstr, 0.5D0 / tri, d2gds2 )
c
c... calculate d2f/dkdsigma = d2gdk
dnedk(1) = -yldpr(9)
dnedk(2) = -yldpr(10)
call rab( d2gds2, nstr, nstr, dnedk, 1, d2gdk )
if ( sw(3) ) then
call primat( d2gds2, nstr, nstr, 'D2GDS2' )
call primat( stiff, nstr, nstr, 'STIFF' )
call privec( d2gdk, nstr, 'D2GDK' )
end if
c
-----
c... Fill matrix [A] with sigma related lines
call invsym( stiff, nstr )
call filma( +1, stiff, nstr )
call uvpws( stiff, d2gds2, nstr*nstr, dl(1), w )
call uvpws( w, d2fds2, nstr*nstr, dl(2), w )
call invsym( stiff, nstr )
call filma( +1, stiff, nstr )
k = 0
do 10 j = 1, nstr
do 10 i = 1, nstr
k = k + 1
a(i,j) = w(k)
10 continue
c
call uvpws( gradg, d2gdk, nstr, dl(1), w )

```



```

C
do 20 i = 1, nstr
  a(i,nstr+1) = w(i)
20 continue
C
do 41 i = 1, nstr
  a(i,nstr+2) = gamma(i)
41 continue
C-----
C... Calculate df1/dk = dfldk
dfldk = uv( gradf, dnedk, nstr )
if ( sw(3) ) then
  call prival( dfldk, 'DF1DK')
end if
C-----
C... Fill matrix [A] with yield function 1 related line
do 50 i = 1, nstr
  a(nstr+1,i) = gradf(i)
50 continue
C
a(nstr+1,nstr+1) = dfldk
C-----
C... Fill matrix [A] with yield function 2 related line
do 55 i = 1, nstr
  a(nstr+2,i) = gradfc(i)
55 continue
C
a(nstr+2,nstr+2) = dfdk
C-----
C
if ( sw(3) ) call primat( a, nx, nx, 'A' )
C
end
C
DOUBLE PRECISION FUNCTION DASUM(N,DX,INCX)
C.....
C
TAKES THE SUM OF THE ABSOLUTE VALUES.
C
DOUBLE PRECISION DX(1),DTEMP
INTEGER I,INCX,M,MPL,N,NINCX
C
DASUM = 0.0D0
DTEMP = 0.0D0
IF(N.LE.0)RETURN
IF(INCX.EQ.1)GO TO 20
C
CODE FOR INCREMENT NOT EQUAL TO 1
C
NINCX = N*INCX
DO 10 I = 1,NINCX,INCX
  DTEMP = DTEMP + DABS(DX(I))
10 CONTINUE
DASUM = DTEMP
RETURN
C
CODE FOR INCREMENT EQUAL TO 1
C
C
CLEAN-UP LOOP
C
20 M = MOD(N,6)
IF( M .EQ. 0 ) GO TO 40
DO 30 I = 1,M
  DTEMP = DTEMP + DABS(DX(I))
30 CONTINUE
IF( N .LT. 6 ) GO TO 60
40 MPL = M + 1
DO 50 I = MPL,N,6
  DTEMP = DTEMP + DABS(DX(I)) + DABS(DX(I + 1)) +
DABS(DX(I + 2))
  * + DABS(DX(I + 3)) + DABS(DX(I + 4)) + DABS(DX(I + 5))
50 CONTINUE
60 DASUM = DTEMP
RETURN
END
C
SUBROUTINE FILMA( KOD, A, N )
C.....
C
TO FILL THE NON-FILLED HALF OF A SYMMETRIC MATRIX
C
C... KOD.GT.0 - LOWER HALF IS FILLED
C
C... KOD.LT.0 - UPPER HALF IS FILLED
C.....
C
INTEGER KOD, N
DOUBLE PRECISION A(N,N)
INTEGER I, J
C
IF ( KOD .GT. 0 ) THEN
C... FILL LOWER HALF
DO 15 I=1,N

```

```

DO 10 J=I,N
  A(J,I)=A(I,J)
10 CONTINUE
15 CONTINUE
ELSE IF ( KOD .LT. 0 ) THEN
C... FILL UPPER HALF
DO 25 I=1,N
  DO 20 J=I,N
    A(I,J)=A(J,I)
20 CONTINUE
25 CONTINUE
END IF
END
C
SUBROUTINE FNCEXP( FVAL0, XULT, X, FVAL, GRAD )
C
C.....
C.....
C
DOUBLE PRECISION FVAL0, XULT, X, FVAL, GRAD
C
FVAL = FVAL0 * EXP( -X / XULT )
GRAD = -FVAL / XULT
C
RETURN
END
C
SUBROUTINE INVSYM( A, NA )
C
C.....
C... INVERSE OF A(NA,NA) . A'S LOWER TRIANGLE IS FILLED.
C... IN THE UPPER TRIANGLE WILL BE THE INVERSE.
C... THE LOWER TRIANGLE IS DESTROYED.
C...
C.....
C
DOUBLE PRECISION EPS
PARAMETER ( EPS=1.D-10 )
C
INTEGER NA
DOUBLE PRECISION A(1), D, F, DM
LOGICAL ROWONE
C
INTEGER IHD, N, JRLBEG, IRY, JBEG, IHJ, IHJUP,
JRLEND, IRLB, JRY, IRL, JRL, K, IRL,
IRH, JE,
JRH, JH, KE, NMI
C
C... CHECK IF THIS IS A CORRECT CALL ....
IF ( NA .LT. 1 ) RETURN
IF ( NA .EQ. 1 ) THEN
  A(1) = 1.0D0/A(1)
  RETURN
END IF
C
DM = 0.D0
IHD=0
N=NA
NMI=N-1
JRLBEG=-NMI
ROWONE=.TRUE.
DO 60 IRY=1,NMI
  IHD=IHD+IRY
  D = A(IHD)
  DM = MAX( ABS( D ), DM )
  IF( ABS( D ) .LT. EPS*DM ) then
    print*, 'Cannot invert an ill-conditioned matrix'
    stop
  end if
  D=1.D0/D
  A(IHD)=D
C
JBEG=IHD+1
IHJ=IHD+N
IHJUP=IRY
IHD=JBEG+NMI-IRY
JRLBEG=JRLBEG+N
JRLEND=JRLBEG+IRY-2
IRLB=JRLBEG
DO 40 JRY=JBEG, IHD
  F=A(JRY)*D
  IRLB=IRLB+N
  IRL=IRLB
  IF( ROWONE ) GOTO 15
  DO 10 JRL=JRLBEG,JRLEND
    A(IRL)=A(IRL)-F*A(JRL)
    IRL=IRL+1
10 CONTINUE
15 CONTINUE
A(IRL)=-F
DO 20 K=JRY, IHD
  IHJ=IHJ+1
  A(IHJ)=A(IHJ)-F*A(K)
20 CONTINUE

```



```

A(JRY)=F
IHJUP=IHJUP+1
IHJ=IHJ+IHJUP
40 CONTINUE
IF ( ROWONE ) GOTO 50
DO 45 JRL=JRLBEG,JRLEND
A(JRL)=A(JRL)*D
45 CONTINUE
GOTO 60
50 CONTINUE
ROWONE=.FALSE.
60 CONTINUE
D = A(IHD+N)
DM = MAX( ABS( D ), DM )
IF( ABS( D ) .LT. EPS*DM ) then
  print*, 'Cannot invert ill-conditioned matrix'
  stop
end if
D = 1.DO/D
A(IHD+N)=D
IRLE=IHD+N*1
DO 70 IRL=IRLB,IRLE
A(IRL)=D*A(IRL)
70 CONTINUE
C...
C BACK SUBSTITUTION STARTS NOW.
C...
IRH=IHD-N*1
DO 140 IRY=1,N*1
JE=IHD-1
KE=IHD+IRY-1
DO 120 JRH=IRH,JE
JH=JRH
D=0.0D0
DO 100 K=IHD,KE
JH=JH+N
D=D+A(K)*A(JH)
100 CONTINUE
A(JRH)=A(JRH)-D
120 CONTINUE
IHD=JE-N
IRH=IRH-N
140 CONTINUE
RETURN
END
c
SUBROUTINE PRIIVL( IVAL, LABEL )
C
C.....
C...
C... PRINT INTEGER VALUE.
C...
C... METHOD:
C... THE VALUE IS IDENTIFIED BY A 1-6 CHARACTER LABEL WHICH
IS
C... PRINTED IN FRONT.
C...
C... INPUT:
C... ARG.LIST: IVAL -VALUE TO BE PRINTED.
C... LABEL -ONE TO SIX CHARACTERS IN A6-FORMAT.
C...
C.....
C
INTEGER IVAL
CHARACTER*(*) LABEL
C
C... PRINT LABEL AND VALUE.
WRITE ( *, 1 ) LABEL, IVAL
RETURN
C
1 FORMAT ( 1X, A, '=', I6 )
END
c
SUBROUTINE PRIMAT( RMAT, N, M, LABEL )
C
C.....
C...
C... PURPOSE:
C... TO PRINT AN REAL MATRIX ON THE LOGICAL OUTPUT UNIT.
C...
C... METHOD:
C... THE MATRIX IS IDENTIFIED BY A 1-6 CHARACTER LABEL
WHICH IS
C... PRINTED ON TOP.
C... THE MATRIX IS PRINTED ROW BY ROW, IN LINES OF TEN
VALUES,
C... THE VALUES ARE FORMATTED IN A 'E11.3' FIELD.
C... COLUMN- AND ROW-INDICES ARE PRINTED ABOVE AND AT THE
LEFT SIDE.
C... A BLANK LINE IS INSERTED BETWEEN EACH ROW.
C...
C... INPUT:
C... ARG.LIST: RMAT -MATRIX TO BE PRINTED.
C... N -NUMBER OF ROWS.
C... M -NUMBER OF COLUMNS.
C...
C... LABEL -ONE TO SIX CHARACTERS IN A6-FORMAT.
C.....
C
INTEGER M, N
DOUBLE PRECISION RMAT(N,M)
CHARACTER*(*) LABEL
C
C... PRINT LABEL AND COLUMN-INDICES.
WRITE(*,1) LABEL, (J,J=1,M)
C
C... PRINT ROWS, PRECEDED BY ROW-INDEX.
DO 10 I = 1, N
WRITE(LOUT,2) I, (RMAT(I,J),J=1,M)
10 CONTINUE
RETURN
C
1 FORMAT(1X,A,':',10(I8,4X)/(8X,10(I8,4X)))
2 FORMAT(1X,I7,1P,10E12.3/(8X,10E12.3))
END
C
SUBROUTINE PRIVAL( RVAL, LABEL )
C
C.....
C...
C... PRINT REAL VALUE.
C...
C... METHOD:
C... THE VALUE IS IDENTIFIED BY A 1-6 CHARACTER LABEL WHICH
IS
C... PRINTED IN FRONT.
C...
C... INPUT:
C... ARG.LIST: RVAL -VALUE TO BE PRINTED.
C... LABEL -ONE TO SIX CHARACTERS IN A6-FORMAT.
C... /INOUT /: LOUT -OUTPUT LOGICAL UNITNUMBER.
C...
C.....
C
DOUBLE PRECISION RVAL
CHARACTER*(*) LABEL
C
C... PRINT LABEL AND VALUE.
WRITE ( *, 1 ) LABEL, RVAL
RETURN
C
1 FORMAT ( 1X, A, '=', 1P, E12.3 )
END
C
SUBROUTINE PRIVEC( RVEC, N, LABEL )
C
C.....
C...
C... PRINT REAL VECTOR (ARRAY).
C...
C... METHOD:
C... THE VECTOR IS IDENTIFIED BY A 1-6 CHARACTER LABEL
WHICH IS
C... PRINTED IN FRONT.
C...
C... INPUT:
C... ARG.LIST: RVEC -VECTOR TO BE PRINTED.
C... N -NUMBER OF VALUES IN VECTOR.
C... LABEL -ONE TO SIX CHARACTERS IN A6-FORMAT.
C...
C.....
C
INTEGER LIN, LOUT, N
DOUBLE PRECISION RVEC(N)
CHARACTER*(*) LABEL
C
C... PRINT LABEL AND VALUES.
WRITE(*,1) LABEL, RVEC
RETURN
C
1 FORMAT(1X,A,':',1P,10E12.3/(8X,10E12.3))
END
C
SUBROUTINE RAB(A,N,M,B,L,R)
C
C.....
C...
C... PURPOSE : R = A * B
C...
C... METHOD : USE FUNCTION VINPRO TO COMPUTE THE INNER-
PRODUCT OF
C... A ROW OF A AND A COLUMN OF B.
C...

```



```

C...
.
C... REAL DIMENSIONS : A(N,M) B(M,L), R(N,L)
.
C...
.
C.....
C
    INTEGER          N, M, L
    DOUBLE PRECISION A(1), B(1), R(1), VINPRO
    INTEGER          I, IR, J, ML
C
    ML=M*L
    DO 130 I=1,N
        IR=I
        DO 110 J=1,ML,M
            R(IR) = VINPRO( A(I), N, B(J), 1, M )
            IR=IR+N
    110 CONTINUE
    130 CONTINUE
    RETURN
    END
C
    SUBROUTINE RABT( A, N, M, B, L, R )
C
C.....
C...
.
C... T
.
C... PURPOSE : R = A*B
.
C...
.
C... METHOD : CALL FUNCTION VINPRO TO COMPUTE INNER-PRODUCT
OF
C... ROW OF A AND ROW OF B.
.
C...
.
C... REAL DIMENSIONS : A(N,M), B(L,M), R(N,L)
.
C...
.
C.....
C
    INTEGER          N, M, L
    DOUBLE PRECISION A(1), B(1), R(1), VINPRO
    INTEGER          IR, J, I
C
    IR=0
    DO 120 J=1,L
        DO 110 I=1,N
            IR = IR + 1
            R(IR) = VINPRO( A(I), N, B(J), L, M )
    110 CONTINUE
    120 CONTINUE
    RETURN
    END
C
    SUBROUTINE RATBA( A, N, M, B, R, H, K )
C
C.....
C...
.
C... PURPOSE : TRIPLE MATRIX PRODUCT R = A * B * A
C...
C... METHOD : USE FUNCTIONS VINPRO AND UV TO COMPUTE
C...
C... REAL DIMENSIONS : A(N,M), B(N,N), R(K,M), H(N)
C...
C.....
C
    INTEGER          N, M, K
    DOUBLE PRECISION A(1), B(1), H(1), R(1), VINPRO, UV
    INTEGER          MN, IR, J, L, I
C
    CALL RSET( 0.00, R, K*M )
C
    MN = M*N
    IR = 1
    DO 100 J = 1, MN, N
        DO 20 L = 1, N
            H(L) = VINPRO( B(L), N, A(J), 1, N )
    20 CONTINUE
        DO 40 I = 1, MN, N
            R(IR) = R(IR) + UV( H(1), A(I), N )
            IR = IR + 1
    40 CONTINUE
        IR = IR + K - M
    100 CONTINUE
    RETURN
    END
C
    SUBROUTINE RMOVE( RFROM, RTO, N )
C
C.....
C...
.
C... PURPOSE:
C... MOVE N REALS FROM < RFROM > TO < RTO > :
C...
C...
C... N N
C... < RTO > := < RFROM >
C... 1 1
C...
C...
C... METHOD:
C... FORTRAN DO-LOOP
C...
C... INPUT:
C... ARG.LIST: RFROM(1:N), N
C...
C... OUTPUT:
C... ARG.LIST: RTO(1:N)
C...
C.....
C
    INTEGER I, N
    DOUBLE PRECISION RFROM(N), RTO(N)
C
    DO 10 I = 1, N
        RTO(I) = RFROM(I)
    10 CONTINUE
    RETURN
    END
C
    SUBROUTINE RSET( RVAL, RAR, N )
C
C.....
C...
.
C... PURPOSE:
C... INITIALIZE REAL ARRAY:
C...
C...
C... N
C... < RAR > := RVAL
C... 1
C...
C...
C... METHOD:
C... FORTRAN DO-LOOP
C...
C... INPUT:
C... ARG.LIST: RVAL, N.
C...
C... OUTPUT:
C... ARG.LIST: RAR(1:N)
C...
C.....
C
    INTEGER I, N
    DOUBLE PRECISION RAR(N), RV, RVAL
C
    RV = RVAL
    DO 10 I = 1, N
        RAR(I) = RV
    10 CONTINUE
    RETURN
    END
C
    DOUBLE PRECISION FUNCTION UV( U, V, N )
C
C.....
C...
.
C... INNER PRODUCT:
C...
C...
C... N
C... UV := SIGMA ( < U > * < V > )
C... 1
C...
C...
C... METHOD:
C... FORTRAN DO-LOOP
C...
C... INPUT:
C... ARG.LIST: U(1:N), V(1:N), N
C...
C... OUTPUT:
C... FUNC.VAL: UV
C...
C.....
C
    INTEGER          N
    DOUBLE PRECISION R, U(N), V(N)
    INTEGER          I
C
    R = 0.00
    DO 10 I = 1, N
        R = R + U(I) * V(I)
    10 CONTINUE
    UV = R

```





```

RETURN
END

C
SUBROUTINE UVMW( V, W, N, U )
C
C.....
C...
C... PURPOSE:
C...   SUBTRACT TWO VECTORS:
C...
C...       N           N           N
C...   < U > := < V > - < W >
C...       1           1           1
C...
C... METHOD:
C...   FORTRAN DO-LOOP
C...
C... INPUT:
C...   ARG.LIST: V(1:N), W(1:N), N
C...
C... OUTPUT:
C...   ARG.LIST: U(1:N)
C.....
C
INTEGER          N
DOUBLE PRECISION U(N), V(N), W(N)
INTEGER          I

C
DO 10 I = 1, N
  U(I) = V(I) - W(I)
10 CONTINUE
RETURN
END

C
SUBROUTINE UVPW( V, W, N, U )
C
C.....
C...
C... PURPOSE:
C...   ADD TWO VECTORS:
C...
C...       N           N           N
C...   < U > := < V > + < W >
C...       1           1           1
C...
C... METHOD:
C...   FORTRAN DO-LOOP
C...
C... INPUT:
C...   ARG.LIST: V(1:N), W(1:N), N
C...
C... OUTPUT:
C...   ARG.LIST: U(1:N)
C.....
C
INTEGER          N
DOUBLE PRECISION U(N), V(N), W(N)
INTEGER          I

C
DO 10 I = 1, N
  U(I) = V(I) + W(I)
10 CONTINUE
END

C
SUBROUTINE UVS( V, N, S, U )
C
C.....
C...
C... PRODUCT OF VECTOR AND SCALAR:
C...
C...       N           N
C...   < U > := < V > * S
C...       1           1
C...
C... METHOD:
C...   FORTRAN DO-LOOP
C...
C... INPUT:
C...   ARG.LIST: V(1:N), N, S
C...
C... OUTPUT:
C...   ARG.LIST: U(1:N)
C.....
C
INTEGER          N
DOUBLE PRECISION S, U(N), V(N)
INTEGER          I

C
DO 10 I = 1, N
  U(I) = V(I) * S
10 CONTINUE

```

```

END

C
SUBROUTINE Drapex( SE, stiff, NSTR, yldpr, sig, depsp )
C
C.....
C...
C... PURPOSE:
C...   TO CALCULATE THE CONSISTENT TANGENT OF THE
C...   'ORTHOTROPIC RANKINE' MODEL
C...   IN THE APEX - Version 2
C...
C.....
C
C... EXTERNALS
C...   DOUBLE PRECISION UV
C
C... PARAMETERS
C...   DOUBLE PRECISION TOLER
C...   PARAMETER      ( TOLER = 1.D-12 )
C...   INTEGER        MSTR
C...   PARAMETER      ( MSTR=6 )
C
C... COMMON DEFINITION
C...   LOGICAL SW
C...   COMMON /SWITCH/ SW(6)
C
C... ARGUMENTS
C...   INTEGER        NSTR
C...   DOUBLE PRECISION SE(NSTR,*), stiff(NSTR,*)
C
C... LOCAL VARIABLES
C...   DOUBLE PRECISION SIG(*)
C...   DOUBLE PRECISION w((mstr+1)*(mstr+1)), dnedk(mstr),
C...   yldpr(*),
C...   $              a(mstr+1,mstr+1), uni(mstr*mstr),
C...   q(mstr*mstr),
C...   $              DEPSP(*), dkdep(mstr), pi(mstr),
C...   $              u(mstr), aa(MSTR,MSTR), bb(MSTR,MSTR),
C...   $              qdepdp(mstr)
C...   INTEGER        IERR, IDUM(mstr+1), i, j, k
C...   DATA IDUM / 1, 2, 3, 4, 5, 6, 7 /
C
C   call rset( 0.d0, dnedk, nstr )
C   call rset( 0.d0, pi, nstr )
C   call rset( 0.d0, a, (mstr+1)*(mstr+1) )
C
C   if ( sw(3) ) CALL PRIVEC( DEPSP, nstr, 'DEPSP ' )
C   if ( sw(3) ) CALL PRIVEC( SIG, nstr, 'SIG ' )
C
C...   Fill unimax
C...   call unimax( uni, nstr )
C...   IF ( SW(3) ) then
C...     CALL PRIMAT( stiff, nstr, nstr, 'stiff' )
C...   end if
C-----
C...   FILL MATRIX A WITH SIGMA RELATED LINES
C...   k = 0
C...   do 10 j = 1, nstr
C...     do 10 i = 1, nstr
C...       k = k + 1
C...       a(i,j) = uni(k)
C...   10 continue
C
C...   dnedk(1) = yldpr( 9 )
C...   dnedk(2) = yldpr(10)
C...   call fabra( aa, bb, nstr, stiff )
C...   call rab( aa, nstr, nstr, dnedk, 1, w )
C...   do 30 i = 1, nstr
C...     a(i,nstr+1) = -w(i)
C...   30 continue
C
C-----
C...   Calculate d(deltak)/depsp = dkdep
C...   CALL fpraor( q, NSTR, 0.d0 )
C...   do 40 i = 4, nstr
C...     q((i-1)*nstr+i) = 0.5d0 / yldpr(4)
C...   40 continue
C...   pi(1) = 1.d0
C...   pi(2) = 1.d0
C...   call rab( q, nstr, nstr, DEPSP, 1, qdepdp )
C...   if ( abs( uv( DEPSP, qdepdp, nstr ) ) .lt. 1.d-16 ) then
C...     call uvs( pi, nstr, 0.5d0, dkdep )
C...   else
C...     call uvs( qdepdp, nstr, 0.5d0 /
C...       $      sqrt( 0.5 * abs( uv( DEPSP, qdepdp, nstr ) ) ),
C...     dkdep )
C...     call uvpws( dkdep, pi, nstr, 0.5d0, dkdep )
C...   end if
C
C...   IF ( SW(3) ) then
C...     CALL PRIVEC( dkdep, nstr, 'dkdep' )
C...   end if
C-----
C...   FILL MATRIX A WITH SOFTENING RELATED LINE

```



```

call invsym( stiff, nstr )
call filma( +1, stiff, nstr )
call rab( stiff, nstr, nstr, dkdep, 1, w )
do 60 i = 1, nstr
  a(nstr+1,i) = w(i)
60 continue
c
a(nstr+1,nstr+1) = 1.d0
c
-----
c
IF ( SW(3) ) CALL PRIMAT( A, nstr+1, nstr+1, 'A' )
c
CALL INVPG( A, nstr+1, IDUM, W, TOLER, IERR )
IF ( IERR .LT. 0 ) then
  print*, 'INVERSION SYSTEM "A" FAILED'
  stop
end if
IF ( SW(3) ) THEN
  CALL PRIMAT( A, nstr+1, nstr+1, 'A-1' )
END IF
c
do 90 i = 1, nstr
  u(i) = dkdep(i)
90 continue
c
k = 0
do 80 i = nstr+1, nstr+1
  do 80 j = 1, nstr
    k = k + 1
    w(k) = a(j,i)
80 continue
c
IF ( SW(3) ) CALL PRIMAT( w, nstr, 1, 'A-1m*1' )
c
call rab( w, nstr, 1, u, nstr, se )
IF ( SW(3) ) CALL PRIMAT( SE, NSTR, NSTR, 'SE-TAN' )
c
call invsym( stiff, nstr )
call filma( +1, stiff, nstr )
c
k = 0
do 85 i = 1, nstr
  do 85 j = 1, nstr
    k = k + 1
    w(k) = a(j,i)
85 continue
c
IF ( SW(3) ) CALL PRIMAT( w, nstr, nstr, 'A-1m*m' )
c
call rab( w, nstr, nstr, bb, nstr, aa )
call rab( aa, nstr, nstr, stiff, nstr, w )
call uvpw( se, w, nstr*nstr, se )
c
Avoid a zero stiffness matrix in the z direction
if ( se(3,3) .eq. 0.d0 ) se(3,3) = 1.d-7
c
IF ( SW(3) ) CALL PRIMAT( SE, NSTR, NSTR, 'SE-TAN' )
c
END
c
SUBROUTINE UVPWS( V, W, N, S, U )
c
C.....
C...
C...      N      N      N
C...      < U > := < V > + < W > * S
C...      1      1      1
C...
C... METHOD:
C...   FORTRAN DO-LOOP
C...
C... INPUT:
C...   ARG.LIST: V(1:N), W(1:N), N, S
C...
C... OUTPUT:
C...   ARG.LIST: U(1:N)
C...
C.....
C...
C...   INTEGER      N
C...   DOUBLE PRECISION S, U(1), V(1), W(1)
C...   INTEGER      I
c
IF ( N .LE. 0 ) RETURN
DO 10 I = 1, N
  U(I) = V(I) + W(I) * S
10 CONTINUE
RETURN
END
c
DOUBLE PRECISION FUNCTION VINPRO( U, IU, V, IV, N )
c
C.....
C...
C...   INNER PRODUCT OF TWO VECTORS:
C...
C...      N,IU      N,IV
C...   VINPRO := SIGMA ( < U > * < V > )
C...      1      1
C...
C...   IU, AND IV ARE INDEX INCREMENTS FOR <U> AND <V>, IF
C...   BOTH
C...   IU AND IV ARE 1, BETTER USE FUNCTION UV.
C...
C... METHOD:
C...   FORTRAN DO-LOOP
C...
C... INPUT:
C...   ARG.LIST: U(.), IU, V(.), IV, N
C...
C... OUTPUT:
C...   FUNC.VAL: VINPRO
C...
C.....
C...
C...   INTEGER      IU, IV, N
C...   DOUBLE PRECISION R, U(1), V(1)
C...   INTEGER      NU, JV, I
c
R = 0.D0
NU = N*IU
JV = 1
DO 10 I = 1, NU, IU
  R = R + U(I) * V(JV)
  JV = JV + IV
10 CONTINUE
VINPRO = R
END
c
C.....
C...
C... PURPOSE:

```